# INTERNATIONAL STANDARD



Second edition 2 017 -02

# Ships and marine technology — Manoeuvring of ships —

# Part 6: Model test specials

Navires et technologie maritime — Manoeuvres des navires — Partie 6: Spécificités des essais sur modèle



Reference number ISO 13643-6:2017(E)



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# **Contents**



#### Foreword Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

The procedures used to develop this document and those intended for its further maintenance are described in the ISO/IEC Directives, Part 1. In particular the different approval criteria needed for the different types of ISO documents should be noted. This document was drafted in accordance with the editorial rules of the ISO/IEC Directives, Part 2 (see www.iso.org/directives).

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For an explanation on the meaning of ISO specific terms and expressions related to conformity assessment, as well as information about ISO's adherence to the World Trade Organization (WTO) principles in the Technical Barriers to Trade (TBT) see the following URL: www.iso.org/iso/foreword.html.

The committee responsible for this document is ISO/TC 8, Ships and marine technology, Subcommittee SC 6, Navigation and ship operations.

This second edition cancels and replaces the first edition (ISO 13643-6:2013), of which it constitutes a minor revision with the following changes:

- in  $3.6$  xy $\psi$ -carriage was inserted;
- in [Tab le 1](#page-5-0) "DNDPDYS " row, the symbo l was changed from "Nϕdyn" to "N′ϕdyn";
- in  $Table 1$  "DYDVTS" row, the SI-unit was changed from " $-$ " to "1";
- in [Equation \(20\)](#page-32-0) part in Equation (20)  $\frac{1}{2}$  p (γ) was changed to  $\frac{1}{2}$  p (γ)
- in  $7.3$  paragraph 3, "moments" was changed to "motions".

A list of all parts in the ISO 13643 series can be found on the ISO website.

# <span id="page-4-0"></span>Ships and marine technology — Manoeuvring of ships —

#### Part 6: — <del>— — — —</del> — Model test specials

# 1 Scope

This document defines symbols and terms and provides guidelines for the conduct of tests to determine the hydrodynamic forces and moments due to prescribed motions under a planar-motion, a circularmotion or an oblique towing or flow system for models of surface ships and submarines. It also defines symbols and terms and provides guidelines for the conduct of tests in a wind tunnel. It is intended to be read in conjunction with ISO 13643-1.

#### $\overline{2}$ ========================

The following documents are referred to in the text in such a way that some or all of their content constitutes requirements of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

ISO 13643-1, Ships and marine technology — Manoeuvring of ships — Part 1: General concepts, quantities and test conditions and test condition s

## 3 Terms and definitions

For the purposes of this document, the following terms and definitions apply.

ISO and IEC maintain terminological databases for use in standardization at the following addresses:

- IEC Electropedia: available at http://www.electropedia.org/
- ISO Online browsing platform: available at http://www.iso.org/obp

#### $3.1$ 3 .1

### planar motion test

manoeuvring test to determine the hydrodynamic forces and moments as functions of lateral velocity and acce leration as well as of angular velocity and acceleration about the z-axis or the y-axis, respectively

#### 3 .2

manoeuvring test to determine the hydrodynamic forces and moments as a function of the angular velocity for surface ships primarily about the z-axis, for submarines primarily about the z-axis, as well as the y-axis

#### 3 .3

#### oblique towing or flow test

manoeuvring test to determine the forces and moments as a function of the drift angle and of the manoeuvring device angle and, in the case of submarines, the angle of attack and hydroplane deflections, in a towing tank, a circulating water tunnel, or a wind tunnel

### <span id="page-5-0"></span>3 .4

#### wind tunnel test

test to determine the aerodynamic forces and moments acting upon the above-water portion of the ship as a function of the relative wind

Note 1 to entry: A wind tunnel may also be used for the underwater hull.

### 3 .5

### manoeuvring device

rudder, azimuthing thruster, hydroplane, cycloidal propeller or equivalent system used to manoeuvre a vessel . . . . . . .

#### 3 .6

#### xyψ-carriage

sub carriage (secondary towing system) to the towing carriage that allows a differential longitudinal, a transverse and a rotational motion of the model in the horizontal plane

# 4 Test-related physical quantities

Test-related physical quantities are according to Table 1. General quantities and concepts are according to ISO 13643-1. <u>to is a set is a</u>



### Table  $1$   $-$  Test-related physical quantities

# ISO 13643 -6 :2017(E)

Symbol	CC-code	SI-unit	Concept	
			<b>Term</b>	Definition or explanation
D	DWI	N	Drag	Force in direction in which relative wind blows
<b>DWL</b>	DWL		Design waterline	(see ISO 13643-1)
FP	FP		Fore perpendicular	(see ISO 13643-1)
$F_{\rm T}$	<b>FTWI</b>	N	Resultant force	$\sqrt{c^2 + b^2}$ and $\sqrt{x^2 + y^2}$ , respectively
$F_{\rm n}$	FN	$\mathbf{1}$	Froude number	(see ISO 13643-1)
$F_{n0}$	FN0	$\mathbf{1}$	(Reference) Froude number	$V_0 / \sqrt{gL}$
GM	GM	m	Metacentric height	(see ISO 13643-1)
$H_{LM}$	<b>HLM</b>	${\rm m}$	Mean height of lateral area above design waterline	$A_{LV}/L_{OA}$
$l_{xx}$	IXX	kg m <sup>2</sup>	Moment of inertia of the model about $x$ -axis	(see ISO 13643-1)
$l_{yy}$	<b>IYY</b>	kg m <sup>2</sup>	Moment of inertia of the model about $v$ -axis	(see ISO 13643-1)
$l_{zx}$	<b>IZX</b>	kg m <sup>2</sup>	Product of inertia of the model	(see ISO 13643-1)
$l_{zz}$	IZZ	kg m <sup>2</sup>	Moment of inertia of the model about z-axis	(see ISO 13643-1)
$\cal K$	MX	N m	Roll moment	Moment about x-axis Relative to ship-fixed axis system
$K_{\phi \text{stat}}$	<b>DKDPST</b>	N m rad-1 a		$\frac{\partial K}{\partial \phi}\mid_{V=0}$ from static test or calculation
$K^\prime$	MXS	$\mathbf{1}$	Non-dimensional roll moment	Especially for submarines: Κ $\stackrel{\rho}{-}{}_{L}^{}3_{V}^{}2$ $\overline{2}$ where $K\left(u,v,w,p,q,r,\,v\,,\,w\,,\,p\,,\,q\,,\,r\,,\phi,\theta\right)$ For surface ships only: К $\frac{\rho}{2}L^3V_0^2$ where $K\left(V_0,\Delta u,v,w,p,q,r,\,v\,,\,w\,,\,p\,,\,q\,,\,r\,,\phi,\theta\right)$
$K_{\text{in}}$ $K_{\rm out}$	<b>MXINS</b> <b>MXOUTS</b>	$\mathbf{1}$ $\mathbf{1}$	In-phase part of non-dimensional roll moment Quadrature part of non-dimensional roll moment	$t+nT$ $\frac{2}{nT}$ $(t)$ sin $\omega t$ dt $t+nT$ $\frac{2}{nT}$ $\cdot$ (t) cos $\omega t$ dt

Table 1 (continued)



# ISO 13643 -6 :2017(E)











represent in the momentum of  $\sim$ 

represent in the function of the function of use  $\alpha$ 

(especially for submarines)

(especially for submarines)

 $u'^2$   $\delta_R^3$ 

Table 1 (continued)

 $M ZUUD3S$  1

MZUVS 1

 $^{N}$ uuδδδ $^{R}$ 

ˆN uv



# ISO 13643 -6 :2017(E)







# ISO 13643 -6 :2017(E)













Symbol	CC-code	SI-unit	Concept	
			<b>Term</b>	Definition or explanation
$\hat{Y_0}$	<b>FY0S</b>	$\,1\,$	Non-dimensional coefficient used in representing $Y'$ when angle of attack $\alpha$ , drift angle $\beta$ , manoeuvring device, and plane angles are zero	
$\hat{Y}^{'}_{\phi}$	<b>FYOPHS</b>	$\mathbf{1}$	Non-dimensional oscillatory lateral motion coefficient	
Z	FZ	${\bf N}$	Normal force	(see ISO 13643-1)
Z'	FZS	$\mathbf{1}$	Non-dimensional normal force	Especially for submarines: $Z_{\rm}$ $\frac{\rho}{\rho}L^2V^2$ 2 where $Z\left(u,v,w,p,q,r,\,v\,,\,w\,,\,\dot{p}\,,\,\dot{q}\,,\,\dot{r}\,,\phi,\theta\right)$ For surface ships only:
				Z $\frac{\rho}{2}L^2V_0^2$ where $Z\left(V_0,\Delta u,v,w,p,q,r,\,v\,,\,w\,,\,p\,,\,q\,,\,r\,,\phi,\theta\right)$
$z_{\rm in}$	<b>FZINS</b>	$\mathbf{1}$	In-phase part of non-dimensional normal force	$t+nT$ $\frac{2}{nT}$ $Z'(t)$ sin $\omega t$ dt
$z_{\rm out}$	<b>FZOUTS</b>	$\mathbf{1}$	Quadrature part of non-dimensional normal force	$t + nT$ $\frac{2}{\sqrt{2}}$ $Z^\prime(t)$ cos $\omega t\,dt$ nT $\boldsymbol{t}$
$\mathcal{A}$ $\boldsymbol{z}_q$	DZDQS	$\,1\,$	Slope through zero of $Z'$ versus $q'$	$\partial Z'$ $\frac{1}{\partial q'}\big _{Z'=Z_0'}$
$z_{\dot q}$	<b>DZDQTS</b>	$\,1\,$		$\partial Z'$ $\frac{1}{\partial \dot{q}'}\big _{Z'= \hat{Z_0'}}$
$\mathcal{L}_{\mathcal{A}}$ $Z_{_W}$	<b>DZDWS</b>	$\,1\,$	Slope through zero of $Z'$ versus $w'$	$\partial Z'$ $\frac{1}{\partial w'}\big _{Z'=Z_0'}$
$z_{\dot{w}}$	<b>DZDWTS</b>	$\,1$		$\partial Z'$ $\frac{1}{\partial \dot{w'}}\big _{Z'=\hat{Z_0'}}$
$\hat{z}^{'}_{pp}$	FZPPS	$\mathbf 1$	Non-dimensional coefficient used in representing $Z'$ as a function of $p'^2$	

Table 1 (continued)

#### **Concept** Symbol | CC-code | SI-unit Term Definition or explanation Non-dimensional coefficient used in FZPRS 1 representing  $Z'$  as a function of  $p'$  r'  $^{\prime}$  pr Non-dimensional coefficient used in  $FZQS$  1 representing  $Z'$  as a function of  $u'$  q'  $^{2}q$ Non-dimensional coefficient used in FZRRS | 1  $^{2}$ rr representing Z′ as a function of  $r^2$  | Non-dimensional coefficient used in FZUUS 1 representing  $Z'$  as a function of  $u'^2$  $^{\prime}$  uu Non-dimensional coefficient used in FZVPS 1 representing  $Z'$  as a function of  $v' p'$  $\omega_{\nu p}$ Non-dimensional coefficient used in **FZVRS** representing  $Z'$  as a function of  $v'$   $r'$  $\frac{2}{r}$ FZVRS <sup>1</sup> Non-dimensional coefficient used in FZWS 1 representing  $Z'$  as a function of  $u' w'$  $\sim_w$ Non-dimensional coefficient used in representing  $Z'$  as a function of FZWQAS <sup>1</sup>  $\omega|_q$  $\cdots$   $\cdots$  $\mathbf{v}$  $\cdots$   $\cdots$  $-$  +  $w$   $\cdots$   $\cdots$ <sup>|</sup> <sup>|</sup> FZWWS <sup>1</sup> representing Z′ as a func t ion of ˆZ ′  $|w'|$   $\sqrt{v}$  +  $w$   $\sqrt{v}$ Non-dimensional coefficient used in representing  $Z^\prime$  as a function of FZWWAS <sup>1</sup>  $\omega_w$  $w' \sqrt{v} + w$ Non-dimensional coefficient used in FZWAS <sup>1</sup>  $\mu$ <sup>L</sup> representing  $Z'$  as a function of  $u'|w'|$ Non-dimensional coefficient used in representing  $Z'$  as a function of FZDBS 1  $^{Z}$  $_{\delta B}$  $u \int_{B}$ Non-dimensional coefficient used in FZDSS 1  $z_{\delta s}$  $r_{\rm c}$  is a function of  $u - v_{\rm s}$ Non-dimensional coefficient used in representing  $Z'$  when angle of attack  $FZ0S$  1  $\alpha$ , drift angle  $\beta$ , manoeuvring device,  $^2$  0 and plane angles are zero z  $\vert$  Z m Normal position (see ISO 13643-1) Normal position of centre of pres-Normal position of centre of pres-<br>sure  $\begin{array}{|c|c|c|} \hline -K/Y \ \hline \end{array}$ z<sub>F</sub> F ZFO MODELL REPORT OF THE SECOND PROPERTY OF THE SECOND MUSIC CONTINUES OF THE SECOND MUSIC CONTINUES OF THE

<span id="page-22-0"></span>

Symbol	CC-code	SI-unit	Concept		
			<b>Term</b>	<b>Definition or explanation</b>	
$Z_{\rm G}$	ZG	m	Normal position of centre of gravity of the model	(see ISO 13643-1)	
$z_{0A}$	Z0A	m	Reference height		
$\alpha$	ALFA	rada	Angle of attack	(see ISO 13643-1)	
$\beta$	<b>BET</b>	rada	Drift angle	(see ISO 13643-1)	
$\Delta u$	<b>DVX</b>	$m s-1$	Surge velocity	$u - u_0$	
$\Delta u'$	<b>DVXS</b>	$\mathbf{1}$	Non-dimensional surge velocity	$\Delta u/V_0$	
$\delta_{\rm B}$	ANB	rada	Bow plane angle	(see ISO 13643-1)	
$\delta_{\mathsf{R}}$	ANRU	rada	Manoeuvring device angle	(see ISO 13643-1)	
$\delta_{\rm S}$	ANS	rada	Stern plane angle	(see ISO 13643-1)	
$\theta_{\rm S}$	<b>TRIMS</b>	rada	Trim angle	(see ISO 13643-1)	
$\theta_0$	THOPMM	rada	Amplitude of pitch oscillation	$\overline{\phantom{0}}$	
$v_A$	<b>VKAI</b>	$m^2s^{-1}$	Kinematic viscosity of air		
$\rho_A$	<b>RHOAI</b>	$kg m-3$	Air density		
$\rho$	<b>RHOWA</b>	$kg m-3$	Water density		
$\phi_{\rm S}$	<b>HEELANG</b>	rada	Heel (bank) angle	(see ISO 13643-1)	
$\phi_0$	PHOPMM	rada	Amplitude of roll oscillation		
$\psi$	<b>PSIH</b>	rada	Heading	(see ISO 13643-1)	
$\psi_{\rm WR}$	PSIWREL	rada	Relative wind direction	(see ISO 13643-1)	
$\psi_0$	<b>PSOPMM</b>	rada	Amplitude of yaw oscillation	$\overline{\phantom{0}}$	
$\omega$	<b>OMN</b>	$s-1$	Angular velocity	$2 \pi /T$	
For angles, the unit ° (degree) may be used. a b The unit kn, common in navigation, may be used.					

Table 1 (continued)

#### $\overline{5}$ **General test conditions** 5 General test conditions

In addition to the general test conditions outlined in ISO 13643-1, the following specific test conditions shall be complied with.

For submarine model tests, surface and bottom effects shall be excluded by the use of suitable measures.

#### Test 6.1 - Planar motion test 6 6 Test 6 .1 — Planar motion test

## 6.1 General

The general test conditions outlined in ISO 13643-1 and Clause 5 shall be complied with.

Generally, the ship model is fixed to the planar motion mechanism by suitable force and torque gauges. For surface ship manoeuvring simulation in only three degrees of freedom  $(x, y, \psi)$ , ensure that the ship model is free to trim, heave, and possibly heel.

In the case of submarines, usually two struts oscillating vertically are used to tow and oscillate the model. Two modes of attachment are used for the tests designated as vertical-plane orientation and horizontal-plane orientation. For the vertical-plane orientation, either one strut is attached to the upright model through the sail or two struts are attached to the inverted model to avoid interference between the struts and the sail. For the horizontal-plane orientation, the model is rotated  $90^\circ$  and the struts are usually attached through its side. Support by one strut from the aft is also possible.

During sway tests, and submarine roll tests, the following data shall be measured:

- moment about x-axis  $K$  (during sway tests, only if the model is restrained in heel);
- moment about  $z$ -axis  $N$ ;
- longitudinal force  $X$ ;
- $\text{lateral force}$   $Y;$

and for submarines in tests about  $\nu$ -axis:

- moment about  $y$ -axis  $M$ ;
- $normal force$   $Z$ .

In surface sharp and the Froude number of surface sharped  $\mathcal{L}_f$  ,  $\mathbf{H0}$  , where such a large waves , shall less the influence  $\mathcal{L}_f$  surface be identica life and function and function  $\mu$  is the effect the effect the effect the effect the effect of  $\mu$  is cannot it be matched; it shall be ensured that the scale instrument function instrument function  $\mathcal{C}$  in  $\mathcal{C}$  in  $\mathcal{C}$ Turbulence stimulators near the bow can be used as necessary.

Since the control surface(s) and propeller(s) affect the coefficients, both should be implemented in the model. For specific problems, tests may be run with bare hulls.

For surface ship models, the model mass, m, corresponds to the displacement volume,  $\nabla$ . For submarine models, the ballast should be adjusted both in quantity and location to establish a condition of neutral buoyancy and level trim. It is difficult to establish an exact condition of neutral buoyancy. Therefore, the exact model mass condition (negative or positive buoyancy) is determined by means of an inclining test, where the change of the axial force with the trim angle is equal to the excess of buoyancy.

The s tab instrument is a lowed in submarked in the coordinate interest in the coordinate in the continues of the centre of gravity x<sup>G</sup> and z<sup>G</sup> are determ ined exper imenta l ly by perform ing incl in ing tes ts (s tands ti l ls) or by calculation.

The model model model is interested in  $\Delta \Delta I$  -  $\Delta \Delta I$  and for submarines and from  $\Delta I$   $\Delta I$ tests performed in air. In the case of submarines, all apertures in the model are sealed and those spaces within the model which will subsequently be free flooding in towing tests are filled with water.

In the linear theory of small departures from steady reference motions of submarines and surface ships, it is standard practice to employ the idea of hydrodynamic derivatives. These derivatives permit the magnitudes of fluid forces and moments to be specified. The derivatives referred to in the maritime literature have invariably been "slow motion derivatives" which serve to determine the vessel's hydrodynamic stability for small motions about  $y$ - and  $z$ -axes.

The theory of the planar motion technique is recast in terms of oscillatory coefficients since they are more appropriate for use where the planar motion mechanism is concerned. Oscillatory coefficients are frequency dependent. If the frequency of the oscillatory motion is made very small, they approximate slow motion derivatives.

The planar motion mechanism (PMM) is essentially a device for oscillating a ship or submarine model while being towed in a tank. The mechanism allows separating the motions of a body moving through a fluid into the pure rotations about  $y$ - (or  $z$ -) axis and pure translatory motions in the direction of  $z$ - (or  $y$ -) axis. Combined motions can also be generated. The differential equations of motion referred to a moving body axis system are used to establish a direct and explicit relationship between the various rotary and acceleration derivatives and the measured in-phase and quadrature parts of the forces and moments. The linear force and moment equations describing the body motions with respect to the initial equilibrium conditions can be written as:

<span id="page-24-0"></span>Lateral force:

$$
Y = \left(\frac{\rho L^4}{2}Y_{\dot{r}}' - m x_{\dot{G}}\right)\dot{r} + \left(\frac{\rho L^4}{2}Y_{\dot{p}}' + m z_{\dot{G}}\right)\dot{p} + \frac{\rho L^3}{2}Y_{\dot{p}}'V_0 p + \left(\frac{\rho L^3}{2}Y_{\dot{r}}' - m\right)V_0 r + \left(\frac{\rho L^3}{2}Y_{\dot{v}}' - m\right)\dot{v} + \frac{\rho L^2}{2}\left(Y_{\dot{v}}'V_0 v + Y_0' V_0^2 + Y_{\phi \text{dyn}}' V_0^2 \phi\right) + Y_{\phi \text{stat}}\phi
$$
\n(1)

Moment about x-axis (if model is restrained in heel during sway tests or for roll tests):

$$
K = \left(\frac{\rho L^5}{2} K_{\dot{p}}' - I_{xx}\right) \dot{p} + \left(\frac{\rho L^5}{2} K_{\dot{r}}' + I_{zx}\right) \dot{r} + \frac{\rho L^4}{2} K_{p}' V_{0} p + \left(\frac{\rho L^4}{2} K_{r}' - m z_{G}\right) V_{0} r + \left(\frac{\rho L^4}{2} K_{\dot{v}}' + m z_{G}\right) \dot{v} + \frac{\rho L^3}{2} \left(K_{v}' V v + K_{0}' V_{0}^2 + K_{\phi \text{dyn}}' V_{0}^2 \phi\right) + K_{\phi \text{stat}} \phi
$$
\n(2)

Moment about z-axis:

$$
N = \left(\frac{\rho L^5}{2} N_{\dot{r}}^{\prime} - I_{zz}\right) \dot{r} + \left(\frac{\rho L^5}{2} N_{\dot{p}}^{\prime} + I_{zx}\right) \dot{p} + \frac{\rho L^4}{2} N_{p}^{\prime} V_{0} p + \left(\frac{\rho L^4}{2} N_{r}^{\prime} - m_{x}{}_{G}\right) V_{0} r + \left(\frac{\rho L^4}{2} N_{\dot{v}}^{\prime} - m_{x}{}_{G}\right) \dot{v} + \frac{\rho L^3}{2} \left(N_{v}^{\prime} V_{0} v + N_{0}^{\prime} V_{0}^{2} + N_{\phi \text{dyn}}^{\prime} V_{0}^{2} \phi\right) + N_{\phi \text{stat}} \phi
$$
\n(3)

For submarines, the following additional equations are used.

Normal force: . . <u>. . . . . . . . . .</u> . . . .

$$
Z = \left(\frac{\rho L^4}{2} Z_{\dot{q}}' + m x_G\right) \dot{q} + \left(\frac{\rho L^3}{2} Z_{\dot{w}}' - m\right) \dot{w} + \left(\frac{\rho L^3}{2} Z_{\dot{q}}' + m\right) V_0 q + \frac{\rho L^2}{2} \left(Z_{\dot{w}}' V_0 w + Z_0' V_0^2\right)
$$
(4)

Moment about y-axis:

$$
M = \left(\frac{\rho L^5}{2} M_q' + I_{yy}\right) \dot{q} + \left(\frac{\rho L^4}{2} M_w' + m x_G\right) \dot{w} + \left(\frac{\rho L^4}{2} M_q' - m x_G\right) V_0 q + \frac{\rho L^3}{2} \left(M_w' V_0 w + M_0' V_0{}^2\right) + M_{\theta \text{stat}}\tag{5}
$$

Non linear dependencies or their respective derivatives shall be taken into account, especially in the case of numerical simulation of arbitrary manoeuvres employing suitable mathematical simulation algorithms. Terms which do not involve accelerations or angular velocities (in the equations of motion quotea *r*<sub>0</sub> ),  $r_{vw}$  $_{\prime\prime\prime}$ , and  $_{\rm r_{\rm v}|_V}$  $_{\perp}$  ) are obtained both in character and values directly and more reliably from an oblique towing or flow test (see *Clause 8*). Terms which involve angular velocities other than angular motion about x-axis (in this case  $\bm{r}_{\textit{vq}}$  $vq \cdot {^Yv}|_r$  $\vert r\vert$ <sup>,</sup>  $^{\prime}$  wr  $_{\rm\scriptscriptstyle{W}}$ ,  $^{Y}$  qr  $_{qr}$ , and  $_{r|r}$  $_1$  ) are obtained both in character and value directly from a circular motion test (see *Clause 7*).

NOTE The type of nonlinear derivatives or coefficients largely depends on the mathematical simulation algorithm to be used.

#### 6 .2 Description

After adjusting the specific test parameters, such as

— d is defined and d it use the model model model model model model model model  $\mu$  ,  $\mu$  () of the oscillation about a model mod z-axis or the amp l itude , ϕ0, of the osc i l lation about x-axis , and

<span id="page-25-0"></span>oscillation period, T, with corresponding angular velocity,  $\omega$ 

the towing carriage carrying the planar motion mechanism is moved at a given constant forward speed, V0, with the p lanar motion mechan ism superimpos ing the given per iod ic motion on th is movement . The propeller revolutions are to be set according to the corresponding self-propulsion point of the full-scale ship or the model.

Constant carriage speed can be considered satisfactory for small amplitude oscillations. For larger amplitudes as possible with an  $xy\psi$ -subcarriage, an adequate oscillatory motion in the tank-longitudinal direction may need to be superimposed to make the model speed constant.

It is important that the oscillatory motions of the model are satisfactory approximations to simple harmonics and that the frequency is low enough for a proper approximation of oscillatory coefficients to slow motion derivatives. to s low motion derivatives .

Certain planar motion systems can be driven with more complex trajectories which aim at specific nonlinear terms.

#### 6 .3 Analysis and presentation of results of a planar motion test

#### 6.3.1 Tests in the horizontal plane of motion



Figure  $1$  – Planar motion mechanism with the model in horizontal plane orientation

If a small sinusoidal lateral motion is superimposed to the forward moving model by setting

$$
a(t) = a_0 \sin \omega t
$$
 and  $\psi(t) = \psi_0 = 0$ 

the following derivatives can be determined:

 $\overline{1}$ 

$$
Y'_{\nu} = \frac{m}{\frac{\rho}{2}L^3} + \frac{\partial Y'_{\text{in}}}{\partial \left(\frac{\omega^2 a_0 L}{V_0^2}\right)_{\omega=0}}
$$
(6)  

$$
Y'_{\nu} = -\frac{\partial Y'_{\text{out}}}{\partial \left(\frac{\omega a_0}{V_0}\right)_{\omega=0}}
$$
(7)

(8)

(9)

$$
N_{\nu}^{'} = \frac{m x_G}{\frac{\rho}{2} L^4} + \frac{\partial N_{\text{in}}^{'}}{\partial \left(\frac{\omega^2 a_0 L}{V_0^2}\right)_{\omega=0}}
$$
  

$$
N_{\nu} = -\frac{\partial N_{\text{out}}^{'}}{\partial \left(\frac{\omega a_0}{V_0}\right)_{\omega=0}}
$$

 $\sim$  $\log_{10}$ 

and if the roll moment were measured:

 $\overline{1}$ 

$$
K_{\nu}^{'} = -\frac{m z_{\text{G}}}{2} L^{4} + \frac{\partial K_{\text{in}}^{'}}{\partial \left(\frac{\omega^{2} a_{0} L}{V_{0}^{2}}\right)_{\omega=0}}
$$
(10)  

$$
K_{\nu} = -\frac{\partial K_{\text{out}}^{'}}{\partial \left(\frac{\omega a_{0}}{V_{0}}\right)_{\omega=0}}
$$
(11)

dimensional amplitudes of motion parameters  $\omega^2 a_0 L/V_0^2$  and  $\omega a_0/V_0$ , respectively, which vary  $a_{0}L$  /  ${V_{0}}^{2}\,$  and  $\,\omega$   $a_{0}$  /  $V_{0}$ , respectively, which vary due to consequently and vertex  $\alpha$  is an angular  $\alpha$  and  $\alpha$  ,  $\alpha$  and  $\alpha$  and  $\alpha$  . Let  $\alpha$  and  $\alpha$  and yield the slopes of the curves used to determine the hydrodynamic derivatives.

NOTE Where results are available from an oblique towing or flow test (see Clause 8), these values are to be preferred because these are the simplest, most direct experiments which can be carried out to obtain the derivatives concerned and are inherently more reliable.



Key

- Yin
- $\mathbf{m}$
- $\blacktriangle$

 $n$ 

 $^{\text{X}}$  non-dimensional linear acceleration parameter  $\omega^2$  $a_0 L / V_0$ 

<sup>Y</sup> in-phase parts of non-dimens ional lateral force , ro ll moment, and moment about z-axis coefficients

#### Figure 2 — Variation of in-phase parts of non-dimens ional latera l force , roll moment, and moment and with value  $\sim$  coefficients in the state  $\sim$  and  $\sim$  and  $\sim$  linear  $\sim$   $\sim$   $\sim$   $\sim$   $\sim$   $\sim$   $\sim$

<span id="page-28-0"></span>

Key

- out
- 
- <sup>-</sup> out  $\mathbf{r}$ '

$$
100*K_{\text{out}}
$$

- $^{\rm X}$  non-dimensional linear velocity parameter  $\omega a_0^{\rm}/\nu_{\rm g}$
- <sup>Y</sup> quadrature parts of non-dimens ional lateral force , ro ll moment, and moment about z-axis coefficients 90 ° out of phase with respect to movement

#### Figure 3 — Variation of quadrature parts of non-dimensional lateral force, roll moment, and moment about z-axis coefficients  $(90^\circ$  out of phase with respect to movement) with nondimensional linear velocity amplitude

If a small sinusoidal rotation about z-axis is superimposed on the forward moving model by setting

$$
a(t) = \psi_0 V_0 \cos \omega t
$$
  

$$
\psi(t) = \psi_0 \sin \omega t
$$

the following derivatives can be determined:

 $\overline{\phantom{a}}$ 

$$
Y'_{\dot{r}} = \frac{m x_{\rm G}}{2} + \frac{\partial Y^{'}_{\rm in}}{\partial \left(\frac{\omega^2 \psi_0 L^2}{V_0^2}\right)}_{\omega=0}
$$

 $(12)$ 

$$
Y'_{r} = \frac{m}{\frac{\rho}{2}L^{3}} - \frac{\partial Y'_{out}}{\partial \left(\frac{\omega \psi_{0}L}{V_{0}}\right)} \bigg|_{\omega=0}
$$
\n
$$
N'_{r} = \frac{I_{zz}}{\frac{\rho}{2}L^{5}} + \frac{\partial N'_{in}}{\partial \left(\frac{\omega^{2}\psi_{0}L^{2}}{V_{0}^{2}}\right)} \bigg|_{\omega=0}
$$
\n
$$
N'_{r} = \frac{m x_{G}}{\frac{\rho}{2}L^{4}} - \frac{\partial N'_{out}}{\partial \left(\frac{\omega \psi_{0}L}{V_{0}}\right)} \bigg|_{\omega=0}
$$
\n(15)

$$
K'_{r} = -\frac{I_{zx}}{\frac{\rho}{2}L^{5}} + \frac{\partial K'_{in}}{\partial \left(\frac{\omega^{2}\psi_{0}L^{2}}{V_{0}^{2}}\right)_{\omega=0}}
$$
(16)  

$$
K'_{r} = -\frac{m z_{G}}{\frac{\rho}{2}L^{4}} - \frac{\partial K'_{out}}{\partial \left(\frac{\omega \psi_{0}L}{V_{0}}\right)_{\omega=0}}
$$
(17)

The in-phase and quadrature parts of force and moment are plotted against the corresponding nondimensional amplitudes of motion parameters  $\omega^2$  $\overline{\phantom{0}}$  $\partial^2{\psi}_0 L^2$  /  $V_0^{\phantom{0}2}$  and  $\omega\,{\psi}_0 L$  /  $V_0$ , respectively, which vary due to consequently and vertex  $\mathcal{O}(1)$  ,  $\mathcal{O}(1)$  ,  $\mathcal{O}(1)$  , we are finding to  $\mathcal{O}(1)$  . Let  $\mathcal{O}(1)$  ,  $\mathcal{O}(1)$  ,  $\mathcal{O}(1)$  ,  $\mathcal{O}(1)$  . Let  $\mathcal{O}(1)$  ,  $\mathcal{O}(1)$  ,  $\mathcal{O}(1)$  ,  $\mathcal{O}(1)$  ,  $\mathcal{O}(1$ yield the s lopes of the curves used to determ ine the hydrodynam ic der ivatives .

NOTE Curvature derivatives may be expected to be more accurately obtained from the planar motion test than from the circular motion test (see *Clause 7*) owing to the limited radius of turn if a rotating arm or similar is used.



Nin

Key

 $\bullet$ 

 $n$ '

<sup>X</sup> non-dimensional angular acceleration parameter  $\omega^2$  $\overline{\phantom{a}}$  $^2$ V  $_0$   $L^2$  /V  $_0^2$ 

Y in-phase parts of non-dimensional lateral force, roll moment, and moment about z-axis coefficients

Figure 4 — Variation of in-phase parts of non-dimensional lateral force, roll moment, and moment about z-axis coefficients with non-dimensional angular acceleration amplitude

<span id="page-31-0"></span>

<sup>0</sup> <sup>0</sup> Y quadrature parts of non-dimensional lateral force, roll moment, and moment about z-axis coefficients

#### Figure 5 — Variation of quadrature parts of non-dimensional lateral force, roll moment, and moment about z-axis coefficients (90°out of phase with respect to movement) with nondimensional angular velocity amplitude

### 6 .3 .2 Tests in the vertical plane of motion (for submarines only)



Figure 6 — Planar motion mechanism with the model in vertical plane orientation [model shown in inverted position chosen to avoid interference between sail and mounting strut(s)]

Key

<sup>X</sup>

<span id="page-32-0"></span>If a small sinusoidal motion in z-direction is superimposed to the forward moving model by setting

$$
a(t) = a_0 \sin \omega t
$$
 and  $\theta(t) = \theta_0 = 0$ 

the following derivatives can be determined:

 $\mathbf{r}$ 

$$
Z'_{\dot{w}} = \frac{m}{\frac{\rho}{2}L^3} + \frac{\partial Z'_{\text{in}}}{\partial \left(\frac{\omega^2 a_0 L}{V_0^2}\right)} \tag{18}
$$
\n
$$
Z'_{\dot{w}} = -\frac{\partial Z'_{\text{out}}}{\partial \left(\frac{\omega a_0}{V_0}\right)} \tag{19}
$$
\n
$$
M'_{\dot{w}} = -\frac{m x_{\text{G}}}{2} L^4 + \frac{\partial M'_{\text{in}}}{\partial \left(\frac{\omega^2 a_0 L}{V_0^2}\right)} \tag{20}
$$
\n
$$
M'_{\dot{w}} = -\frac{\partial M'_{\dot{w}}}{\partial \left(\frac{\omega a_0}{V_0}\right)} \tag{21}
$$

The in-phase and quadrature par ts of force and moment are p lotted aga ins t the correspond ing nondimensional amplitudes of motion parameters  $\omega^2 a_0 L / V_0^2$  and  $\omega a_0 / V_0$ , respectively, which vary due to changes of anguna (circuity) w, speed, V0, or amp litude , a0 (see Figures 2 and [3\)](#page-28-0). Hence fit it fits yield the s lopes of the curves used to determ ine the hydrodynam ic der ivatives .

If a small sinusoidal rotation motion about y-axis is superimposed to the forward moving model by setting

$$
a(t) = \theta_0 V_0 \cos \omega t
$$

$$
\theta(t) = \theta_0 \sin \omega t
$$

the following derivatives can be determined:

$$
Z'_{\dot{q}} = -\frac{m x_{\rm G}}{\frac{\rho}{2} L^4} + \frac{\partial Z'_{\rm in}}{\partial \left(\frac{\omega^2 \theta_0 L^2}{V_0^2}\right)_{\omega=0}}
$$
(22)

$$
Z'_{q} = -\frac{m}{\frac{\rho}{2}L^{3}} - \frac{\partial Z'_{\text{out}}}{\partial \left(\frac{\omega \theta_{0}L}{V_{0}}\right)_{\omega=0}}
$$
(23)

$$
\tilde{M}'_{\theta} = -\frac{I_{yy}}{\frac{\rho}{2}L^5} \frac{\omega^2 L^2}{V_0^2} - \frac{M'_{in}}{\theta_0} = \frac{M_{\theta \text{stat}}}{\frac{\rho}{2}L^3V_0^2} - M'_q \frac{\omega^2 L^2}{V_0^2} + M'_q \frac{\omega^4 L^4}{V_0^4} + ...
$$
\n(24)

$$
M_{q}^{'} = \frac{m x_{G}}{2} L^{4} - \frac{\partial M_{\text{out}}^{'}}{\partial \left(\frac{\omega \theta_{0} L}{V_{0}}\right)_{\omega=0}}
$$
(25)

The in-phase and quadrature parts of force and moment are p lotted aga ins t the correspond ing nondimensional amplitudes of motion parameters  $\omega^2\theta_0L^2/V_0^2$  and  $\omega\theta_0L/V_0$ , respectively, which vary <sup>0</sup> <sup>0</sup>  $$ due to changes of angusting versions,  $\mu$  is performance and an proposition  $\mu$  (see Figure 7) . Let  $\mu$  it leaves  $\mu$  is large the slopes of the curves used to determine the hydrodynamic derivatives. A linear curve fit of  $m_{\theta}$ plotted against  $\omega^2L^2/V_0^2$  yields both of the slow motion derivatives  $\frac{1}{\sqrt{2}}$ θ  $\frac{1}{\rho_{13}}$ and  $M_{\theta}$  (see <u>Figure 7</u>).

Mϕs tat shou ld correspond with the va lue found by the tare tes t and shou ld equa l gmGM of the mode l .

<span id="page-34-0"></span>

$$
X = \omega^2 L^2 / V_0^2
$$

<sup>Y</sup> non-dimensional oscillatory coefficient about *y*-axis  $^{\prime\prime\prime}{}_{\theta}$ 

<sup>a</sup> ... . . . . . . length scale of a length scale of an interest of a scale of a <sup>−</sup> ′  $\mathsf I$  iengul scale of  $\mathsf{w}_i$  $\sim$  $\frac{1}{a}$  engin scale of  $M_{\theta}$  $\omega^2 L^2$  $\overline{\phantom{0}}$  $/V<sub>0</sub>$ . .  $\overline{\phantom{a}}$  n.

Figure  $\beta \to \mathsf{Plot}$  of non-dimensional oscillatory coefficient about  $y$ -axis  $\mathscr{M}_\theta$  against  $\omega^2 L^2/V_0^2$ 

### 6.3.3 Tests for angular motion about  $x$ -axis (roll)



Figure 8 — Planar motion mechanism with a submarine model prepared for the determination of roll coefficients

For roll motion tests, submarine models have to be mounted in the horizontal mode position (see Figure 8). The displacements of the model mountings are held fixed while the model is towed down the tank.

<span id="page-35-0"></span>The imposed rotation is such that

$$
\phi(t) = \phi_0 \sin \omega t
$$

and we get

$$
\dot{Y}_{\phi}^{\prime} - \frac{Y_{\phi \text{stat}}}{\frac{\rho}{2}L^{2}V_{0}^{2}} = \frac{m z_{\text{G}}}{\frac{\rho}{2}L^{4}} \frac{\omega^{2}L^{2}}{V_{0}^{2}} - \frac{Y_{\text{in}}^{\prime}}{\phi_{0}} - \frac{Y_{\phi \text{stat}}}{\frac{\rho}{2}L^{2}V_{0}^{2}} = Y_{\phi \text{dyn}}^{\prime} - Y_{\rho}^{\prime} \frac{\omega^{2}L^{2}}{V_{0}^{2}} + Y_{\rho}^{\prime} \frac{\omega^{4}L^{4}}{V_{0}^{4}} + ...
$$
\n(26)

$$
Y_p' = -\frac{\partial Y_{\text{out}}'}{\partial \left(\frac{\omega \phi_0 L}{V}\right)}
$$
(27)

$$
\tilde{K}_{\phi}^{\prime} - \frac{K_{\phi \text{stat}}}{\frac{\rho}{2}L^{3}V_{0}^{2}} = -\frac{I_{xx}}{\frac{\rho}{2}L^{5}} \frac{\omega^{2}L^{2}}{V_{0}^{2}} - \frac{K_{\text{in}}^{\prime}}{\phi_{0}} - \frac{K_{\phi \text{stat}}}{\frac{\rho}{2}L^{3}V_{0}^{2}} = K_{\phi \text{dyn}}^{\prime} - K_{\rho}^{\prime} \frac{\omega^{2}L^{2}}{V_{0}^{2}} + K_{\rho}^{\prime} \frac{\omega^{4}L^{4}}{V_{0}^{4}} + ...
$$
\n(28)

$$
K_{p}^{'} = -\frac{\partial K_{\text{out}}^{'}}{\partial \left( \frac{\omega \phi_{0} L}{V_{0}} \right)}
$$
(29)

$$
\tilde{N}'_{\phi} - \frac{N_{\phi \text{stat}}}{\frac{\rho}{2} L^3 V_0^2} = \frac{I_{xz}}{\frac{\rho}{2} L^5} \frac{\omega^2 L^2}{V_0^2} - \frac{N'_{\text{in}}}{\phi_0} - \frac{N_{\phi \text{stat}}}{\frac{\rho}{2} L^3 V_0^2} = N'_{\phi \text{dyn}} - N'_{\rho} \frac{\omega^2 L^2}{V_0^2} + N'_{\rho} \frac{\omega^4 L^4}{V_0^4} + \dots
$$
\n(30)

$$
N_p' = -\frac{\partial N_{\text{out}}'}{\partial \left(\frac{\omega \phi_0 L}{V_0}\right)}\tag{31}
$$

Linear curve fits (see  $\tilde{M}_{\theta}$  in Figure 7) of  $\tilde{Y}_{\phi}$  –  $\frac{1}{\sqrt{2}}$  $$ φ  $\phi$  $\phi = \frac{1}{\rho_{12V}}$ ,  $K'$   $\cdot$   $\cdot$ φ φ  $\phi = \frac{1}{\rho_{13}I}$ <sup>3</sup> , and  $N =$  $\cdot$   $\cdot$ φ  $\phi$  $\phi = \frac{1}{\rho_{13}I}$ <sup>3</sup>  $\overline{\phantom{a}}$ **p** – – – – – – –

against  $\omega$ 2 L2/V $_0$ 2 yield the slow motion derivatives  $Y_{\phi \rm dyn}$ ,  $Y_{\dot p}$ ,  $K_{\phi \rm dyn}$ ,  $K_{\dot p}$ ,  $N_{\phi \rm dyn}$ , and  $N_{\dot p}$ . Y $\phi$ stat, K $\phi$ stat, and are to be determined by including the form in independent time  $\alpha$  and by calculation .

### 6 .4 Designation of a planar motion test

<sup>2</sup> <sup>2</sup> <sup>2</sup>

 $\overline{\phantom{0}}$ 

### 6 .4.1 Designation of a planar motion test in the horizontal plane (H)

Designation of a planar motion test in the horizontal plane (H) according to ISO 13643-6 (6), Test 1 (1),  $\frac{1}{2}$  tonducted at a model towing speed  $\frac{1}{0}$  = 3 m s =  $\frac{1}{0}$  , an oscination period T = 6 s  $\frac{1}{0}$  or  $\frac{1}{0}$  a model sway amp litude and and and a second of the group of the second way of the second was provided with the second was

#### Planar motion test ISO 13643 - 6.1 × H/03/06/00/05

#### <span id="page-36-0"></span>6.4.2 Designation of a planar motion test in the vertical plane  $(V)$

Designation of a planar motion test in the vertical plane (V) according to ISO 13643-6 (6), Test 1 (1), conducted at a model towing speed V0 = 3 m s − (03), an oscination period T = 6 s (00), a model neave amp litude and  $\alpha$  it is and  $\alpha$  it is and and a proceed amplitude  $\alpha$  (  $\alpha$  ) is a set of  $\alpha$ 

#### Planar motion test ISO 13643 - 6.1 × V/03/06/10/03

#### 6.4.3 Designation of a planar motion test for roll motion (R)

Designation of a planar motion test for roll motion (R) according to ISO 13643-6 (6), Test 1 (1), conducted at a model towing speed  $v_0$  = 3 m s  $\overline{\phantom{a}}$  (05), an oscination period T = 6 s (06), and a ron amp is a series  $\mathcal{F} \cup \mathcal{F} = \{ \mathcal{F} \in \mathcal{F} \}$ .

#### Planar motion test ISO 13643 -  $6.1 \times R/03/06/05$

#### Test 6.2 – Circular motion test  $\overline{7}$ 7 Test 6 .2 — Circular motion test

#### 7 .1 General

Besides the general test conditions outlined in ISO 13643-1 and Clause  $\overline{5}$ , the following specific test conditions shall be complied with.

- $-$  The ship model is fixed to the cantilever of the rotating arm or other circular motion device by a suitable force-sensing and towing device. For surface ship manoeuvring simulation in only three degrees of freedom  $(x, y, \psi)$ , ensure that the ship model is free to trim, heave, and possibly heel.
- To a certain extent, circular motions may be generated by means of an xy $\psi$ -carriage in a towing tank.
- In surface sh ip model tes ts , the Froude number, Fn0, wh ich sca les the in fluence of surface waves , shad la be ma interesting in the surface model and function into and fuse interesting . As Reyno local local l scales the effect of viscosity, cannot be matched, it shall be ensured that the scale model attains fully turbu lent flow (supercritical lift). These timested s timulations can be used near the sector as necessary.
- To avoid measuring errors, only one full rotation per test (including the acceleration phase) should be conducted with the rotating arm or other circular motion device.

During the test, the following data are to be measured:

#### For tests with circular motion about z-axis



#### For tests with circular motion about  $\nu$ -axis (for submarines only)

moment about  $y$ -axis  $M$ ; longitudinal force  $X$ ; Z. normal force

norma l force Z.

## <span id="page-37-0"></span>7 .2 Description

After adjustice is the species of the species , such as c in the species radius in the such as completed that i angle , ενώ τους τους ρείς βρίσκεται p lane and δερ, δερ, δερ, τους είχε τους που από το προσελείωσαν στο απόσ path at a cons tant speed V<sup>0</sup> lead ing to the s teady sh ip -fixed veloc ities p , q , r, u , v, w. Ins tead of u, the speed u0 +  $\sim$  100 motions of the used in the equations of motion with a correspond into subject in the sp l i coefficients , under a second as a second in the sequence shall be surface shall be surface shall be surface sha values of the coefficients vary with the Froude number.

The propeller revolutions are to be set according to the corresponding self-propulsion point of the fullscale ship or the model. In order to remove the component due to gauge zero, offset, and buoyancy, the tare value appropriate to the gauge and the attitude of the model is subtracted for the gauge readings, leaving only the component equal and opposite to the hydrodynamic force or moment on the model.

#### Analysis and presentation of results of a circular motion test  $7.3$

After deduction of the inertial effects, the measurement data are mostly plotted non-dimensionally as a function of the parameters angular velocity, drift angles, and/or manoeuvring device angles (see Figures 9 to [11](#page-39-0) tests with circular motion about the z-axis at different drift angles) and in the case of submarine tests with circular motion about the y-axis as a function of angles of attack and/or hydroplane angles.

The non-dimensional coefficients used for general computer simulation studies are determined by curve-fitting the test data with appropriate equations in accordance with standardized methods. Nondimensional curvature and control derivatives are determined from the slope through zero of the nondimensional curves of force or moment versus the different non-dimensional test parameters. These curvature derivatives may be expected to be more accurately obtained from the planar motion test (see Clause  $6$ ) owing to the limited radius of turn available if a rotating arm or equivalent is used. It is found that unless the rotating arm values lie on a straight line when plotted non-dimensionally against the rate of turn, it is impossible to define the slope at zero turn precisely. Acceleration and roll derivatives are available experimentally from the planar motion test only. It should be emphasized that the values of the first order coefficients that are associated with curve fits to the force and moment curves are not necessarily equal to those customarily used for the corresponding stability and control derivatives in the linearized equations of motion. However, the standard notation often used to describe these firstorder coefficients is the same as that used for the corresponding stability and control derivatives. Here, this is taken into account by different symbols and terms.

The nonlinear derivatives shall be taken into account in the case of numerical simulation of arbitrary manoeuvres employing success successive successive and lating successive inter inter inter in the N°r, N°r, N° Z′q, and M′q, together with the respective resu lts of the p lanar motion tes t (see [C lause 6\)](#page-22-0) and/or ob l ique towing or flow test (see Clause 8) serve to determine the vessel's dynamic stability for small motions about z-axis and y-axis. However, the derivatives may also be used in specialized simulation studies such as design studies of automatic course keeping and/or depth keeping control systems, where a simplified mathematical model is entirely adequate.

The type of nonlinear coefficients largely depends on the form of mathematical simulation algorithm to be used. The coefficients listed in Clause  $4$  shall therefore only be regarded as reference data for an appropriate test evaluation. For submarines in deeply submerged condition, the results are independent of the Froude number and forces and moments can be determined as a function of the longitudinal velocity, u. For surface ships, the hydrodynamic coefficients are determined for the Froude number be longing to the modern speed, V0, him instant of u, the surge versity, 2013 in the form of the force and moment equations. Therefore, there are differences in the coefficients sets used for surface ships or submarines which has to be taken into account when choosing coefficients from Table 1.

## 7 .4 Designation of a circular motion test

Designation of a circular motion test according to ISO 13643-6 Test 2, conducted with the reference axis z (2), a moder reference speed V0 = 3 m s = (03), a circular motion radius R = 10 m (10), a drift angle <sup>β</sup> = 10 ° (10) , and a manoeuvring device angle δ<sup>R</sup> = 20 °(20) :

#### Circular motion test ISO  $13643 - 6.2 \times Z/03/10/10/20$

Designation of a circular motion test according to ISO 13643-6 Test 2, conducted with the reference axis y (Y) , a mode l reference speed V<sup>0</sup> = 3 ms–1 (03 ) , a c ircu lar motion rad ius R = 10 m (10) , a tr im angle  $\mathcal{S} \cup \mathcal{S} = \{ \mathcal{S} \cup \mathcal{S} \mid \mathcal$ 

Circular motion test ISO  $13643 - 6.2 \times Y/03/10/05/03/00$ 



Key

- $N'$  non-dimensional moment about z-axis
- $r'$  non-dimensional angular velocity about z-axis





#### Key

- $Y'$  non-dimensional lateral force
- $r'$  non-dimensional angular velocity about z-axis



<span id="page-39-0"></span>

### Key

- $X'$  non-dimensional longitudinal force
- $r'$  non-dimensional angular velocity about z-axis

### Figure  $11$  – Longitudinal force – Angular velocity about z-axis

#### Test  $6.3$  — Oblique towing or flow test 8

### 8.1 General

An oblique towing or flow test consists of a series with variations of drift angle, angle of attack and manoeuvring device angle within the specified ranges.

Besides the general test conditions outlined in ISO 13643-1 and Clause  $\overline{5}$ , the following specific test conditions shall be complied with.

- The ship model is fixed to the towing carriage by a suitable force-sensing and towing device. For surface ship manoeuvring simulation in only three degrees of freedom  $(x, y, \psi)$ , ensure that the ship model is free to pitch, heave, and possibly heel.
- In surface sh ip mode l tes ts , the Froude number, Fn0, wh ich sca les the influence of surface waves , shah s l intantitum a s correct interest and the function of the scale  $\cdots$  into  $\cdots$  is number,  $\cdots$  if  $\mathbf{0}$ the effect the effect of  $\mu$  at the scale of the scale  $\mu$  at the scale flow at the scale (supercreament)  $\mu$ to be ensured. Turbulence stimulators can be used near the bow as necessary.

During the test, the following data shall be measured:

#### For tests with model rotated about z-axis



### <span id="page-40-0"></span>For tests with model rotated about  $\nu$ -axis (for submarines only)

moment about  $v$ -axis  $M$ ; longitudinal force  $X$ ; normal force  $Z_{\cdot}$ 

### 8.2 Description

After adjustment of the specific test parameters, such as drift angle,  $\beta$ , and manoeuvring device angle, δ<sup>R</sup>, and in the case of submar ines , angle of attack, α, and p lane angle δ<sup>B</sup> or δ<sup>S</sup> as wel l , the sh ip model is force -towed on a s tra ight path at a cons tant speed , V0. The propel ler revo lutions are to be set accord ing to the corresponding self-propulsion point of the full-scale ship or the model. In order to remove the component due to "gauge zero", "offset", and "buoyancy", the tare value appropriate to the gauge and the attitude of the model is subtracted for the gauge readings, leaving only the component equal and opposite to the hydrodynamic force or moment on the model.

In the stationary phase of the test, the measuring time should be at least 25 s.

### 8.3 Analysis and presentation of results of an oblique towing or flow test

The measured data are mostly plotted non-dimensionally as functions of the test parameters drift angle  $\beta$  and/or manoeuvring device angles (see Figures 12 to [14](#page-42-0) show results of the drift-angle tests at different manoeuvring device angles as function of the velocity in direction of the y-axis v) and in the case of submarine trim-angle tests angle of attack  $\alpha$  and/or hydroplane angles.

The non-dimensional coefficients used for general computer simulation studies are determined by curve-fitting the test data with appropriate equations in accordance with standardized methods. Nondimensional dynamic stability and control derivatives are determined from the slope through zero of the non-dimensional curves of force or moment versus the different non-dimensional test parameters. These results are preferred compared to planar motion test data (see Clause  $6$ ) because these are the simplest, most direct experiments which can be carried out to obtain the derivatives concerned and are inherently more reliable. It should be emphasized that the values of the first order coefficients that are associated with curve fits to the force and moment curves are not necessarily equal to those cus tomarily used for the corresponding stability and control derivatives in the linearized equation of motion. This is taken into account by different symbols and terms.

The nonlinear coefficients shall be taken into account, in the case of numerical simulation of arbitrary manoeuvres employing success contracted limit in all s internation and limited to let increase the limit  $\mathcal{V}$  is  $\mathcal{V}$ Z′<sup>w</sup>, and M′<sup>w</sup> together with the respec tive resu lts of the p lanar motion tes t (see [C lause 6\)](#page-22-0) and/or c ircu lar motion test (see Clause 7) serve to determine the vessel's dynamic stability for small circular motions about  $z$ - and  $y$ -axes. However, the derivatives may also be used in specialized simulation studies, such as design studies of automatic course keeping and/or depth keeping control systems, where a simplified mathematical model is entirely adequate.

The type of nonlinear coefficients largely depends on the form of the mathematical simulation algorithm to be used. The coefficients listed in  $Table 1$  shall therefore only be regarded as reference data for an appropriate test evaluation. For submarines in deeply submerged condition, the results are independent of the Froude number and forces and moments can be determined as a function of the longitud inal velocity, u. For surface ships, the hydrodynamic coefficients are determined for the Froude number belong ing to the modern speed , V0, and inserted in the surge vecessity,  $\Delta$ u, is not in the surge force and moment equations. Therefore, there are differences in the coefficients sets used for surface ships or submarines which has to be taken into account when choosing coefficients from Table 1.

### 8.4 Designation of an oblique towing or flow test

Designation of an oblique towing test series (T) for a model rotated about the z-axis (Z) according to ISO 13643 -6 (6) , Tes t 3 (3 ) , conduc ted at a model speed V<sup>0</sup> = 3 m s –1 (03 ) , dr ift angles β ranging from 0 ° in steps of  $3^{\circ}$  to +15°(00:03:+15), angles of attack  $\alpha$  ranging from 0° in steps of 0° to 0° (00:00:00) (i. e. no var iation of angles of attacking) , and manueles from mangles angles fR ranging from −30 ° in s teps of 10 ° to  $+30^{\circ}$  (-30:10:+30):

#### Oblique towing test ISO 13643 - 6.3 × Z/03/00:03:+15/00:00:00/-30:10:+30/T

Designation of an oblique flow test series (F) for a model rotated about the z-axis (Z) according to ISO 13643 -6 (6) , Tes t 3 (3 ) , conduc ted at a mode l speed V<sup>0</sup> = 3 m s –1 (03 ) , dr ift angles β ranging from 0 ° in steps of  $3^\circ$  to +15°(00:03:+15), angles of attack  $\alpha$  ranging from 0° in steps of 0° to 0° (00:00:00) (i. e. no var iation of angle of attacking) , and manueles in senger in senger and manging from −30 ° in s teps of 10 ° to +30° (-30:10:+30):

### Oblique flow test ISO 13643 -  $6.3 \times Z/03/00:03:+15/00:00:00/-30:10:+30/F$

**NOTE** For tests with the model rotated about the y-axis (for submarines only), the identifier would be Y instead of Z. The manoeuvring device angle would be replaced by the stern plane and the bow plane angle.



### Key

- $N'$  non-dimensional moment about z-axis
- non-dimensional lateral velocity  $v'$
- $\delta_{\rm R}$ manoeuvring device angle

#### Figure  $12$  – Moment about z-axis - lateral velocity

<span id="page-42-0"></span>

#### Key

- $Y'$  non-dimensional lateral force
- $v'$  non-dimensional lateral velocity
- $\delta_{\rm R}$ manoeuvring device angle



### Figure  $13$  – Lateral force - lateral velocity

# Key

- $X'$  non-dimensional longitudinal force
- $v'$  non-dimensional lateral velocity
- $\delta_{\rm R}$ manoeuvring device angle

# Figure 14 — Longitudinal force – lateral velocity

# <span id="page-43-0"></span>9 Test 6.4 – Wind tunnel test

#### 9.1 General 9 .1 General

Besides the general test conditions outlined in ISO 13643-1 and Clause  $\overline{5}$ , the following specific test conditions shall be complied with.

- The model of the above-water ship is attached over a model trough with an integrated dynamometer to a rotating disk representing the water surface and serving to vary the relative wind angle.
- The wind tunnel test comprises a series of individual runs. The relative wind angles to be tested range from  $0^{\circ}$  to 180° for ships symmetrical to the xz-plane and from  $0^{\circ}$  to 360° for non-symmetrical ships. For a symmetry check, few points between  $180^{\circ}$  and  $360^{\circ}$  may be considered.

During the test, the following data are to be measured:



To ensure that the test results are independent of Reynolds number, the tests shall be repeated at least at two other Reynolds numbers.

### 9.2 Description

The ship model is tested in a homogeneous flow or in a flow having a velocity gradient. As the relative wind is a superposition of a homogeneous flow due to ship speed and the true wind with a gradient tests are typically performed in a homogeneous flow. For tests with a velocity gradient the power law:

$$
\frac{V_{\text{WT}}}{V_{\text{WTA}}} = \left(\frac{z_0}{z_{0\text{A}}}\right)^{\frac{1}{n}}
$$
(32)

where  $n = 10$  may be used as a simple approximation to the logarithmic profile of the true wind over the sea surface . For function search = 10 m usual = −10 m usua light part = 1

### 9.3 Analysis and presentation of results of a wind tunnel test

The measured data are usually plotted non-dimensionally as a function of the relative wind direction,  $\mu$  wind is a figures in the dynamic matrix  $\mu$  is the relative wind vertex and the lateral lateral lateral limits  $\mu$ above waterline , ALV, serve as reference quantities . The drag , D, and the long itune in the late inserve be relative to the transverse pro j ec ted area of the sh ip above waterl ine , AXV.

If the cross force,  $C$ , and the drag,  $D$ , are measured during the tests, the longitudinal force,  $X$ , and the lateral force,  $Y$ , can be obtained from the following relations:

$$
X = C \sin \psi_{\text{WR}} - D \cos \psi_{\text{WR}}
$$
\n
$$
Y = C \cos \psi_{\text{WR}} + D \sin \psi_{\text{WR}}
$$
\n(33)

The coord inates x<sup>F</sup> and z<sup>F</sup> of the pos ition of the pressure centre of wind force may be ca lcu lated from measured moments about x- and z-axes, respectively, according to the following formulae:

<span id="page-44-0"></span>
$$
x_{\rm F} = N / Y \tag{35}
$$

$$
z_{\rm F} = -K / Y \tag{36}
$$

#### 9.4 Designation of a wind tunnel test

Designation of a wind tunnel test according to ISO 13643-6 Test 4, conducted at a wind velocity for the  $\frac{1}{100}$  and simp of VWR = 20 m s = and relative wind directions  $\psi_{\text{WR}}$  ranging from 0 ° in steps of 15 ° to  $180^\circ (000/15/180)$ :

#### Wind tunnel test ISO 13643 - 6.4 × 20/000:15:180



Figure 15 – Cross-force coefficient as function of relative wind direction



Figure  $16$  – Drag coefficient as function of relative wind direction



Figure 17 – Lateral-force coefficient as function of relative wind direction



Figure  $18$  – Coefficient of moment about z-axis as function of relative wind direction



Figure 19 – Roll-moment coefficient as function of relative wind direction

<span id="page-46-0"></span>

Figure & 20 — 20 million-force and all interests and continue and above the ship water of ship above  $\alpha$ function of relative wind direction

ISO 13643 -6 :2017(E)

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