INTERNATIONAL STANDARD

ISO 1275

Fourth edition 2006-07-01

Double-pitch precision roller chains, attachments and associated chain sprockets for transmission and conveyors

Chaînes de précision à rouleaux à pas double, plaques-attaches et pignons dentés correspondants pour transmission et manutention



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Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

International Standards are drafted in accordance with the rules given in the ISO/IEC Directives, Part 2.

The main task of technical committees is to prepare International Standards. Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

Attention is drawn to the possibility that some of the elements of this document may be the subject of patent rights. ISO shall not be held responsible for identifying any or all such patent rights.

ISO 1275 was prepared by Technical Committee ISO/TC 100, Chains and chain sprockets for power transmission and conveyors.

This fourth edition cancels and replaces the third edition (ISO 1275:1995), which has been technically revised.

Introduction

This International Standard has been established to cover a range of chains used in the majority of countries in the world by unifying dimensions, strengths and other data from current national standards.

The principal feature of these chains is their derivation from the ISO 606 series by using the standard round parts in links that are double the standard pitch.

Chains have been adopted from the ANSI, BS and DIN double-pitch series to form a range from 25,4 mm to 101,6 mm pitch. Versions of chains are included with normal and with thicker plate materials, with the alternatives of smaller or larger rollers, as well as a range of attachments and sprockets.

The dimensions of the chains provide for complete interchangeability of individual links, and the sprocket dimensions allow complete interchangeability of chains of the same pitch and the same chain designation (either A or B series).

Double-pitch precision roller chains, attachments and associated chain sprockets for transmission and conveyors

1 Scope

This International Standard specifies requirements for double-pitch precision roller chains suitable for the mechanical transmission of power and for conveyors, together with those for their associated sprockets. It covers dimensions, tolerances, length measurement, preloading and minimum tensile strengths.

These double-pitch chains have been derived from some of the short-pitch transmission precision roller chains covered by ISO 606 having certain common dimensions but of double the pitch.

The chains are intended for use under less onerous conditions with respect to speed and power transmitted than are the base chains from which they are derived.

This International Standard primarily applies to sprockets with 5 to 75 teeth inclusive (with intermediate numbers of teeth 5 1/2 to 74 1/2 inclusive).

The preferred numbers of teeth are 7, 9, 10, 11, 13, 19, 27, 38 and 57.

2 Normative references

The following referenced documents are indispensable for the application of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

ISO 286-2, ISO system of limits and fits — Part 2: Tables of standard tolerance grades and limit deviations for holes and shafts

ISO 606, Short-pitch transmission precision roller and bush chains, attachments and associated chain sprockets

3 Transmission chains

3.1 Assembly and component nomenclature

The nomenclature of the chain assemblies and their component parts is illustrated in Figures 1 and 2.

NOTE The figures do not define the actual form of the chain plates.

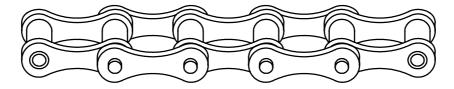
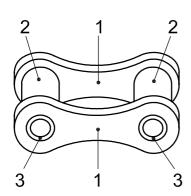


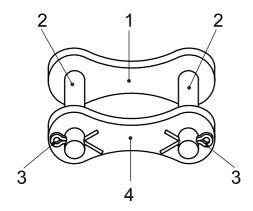
Figure 1 — Chain assembly



Key

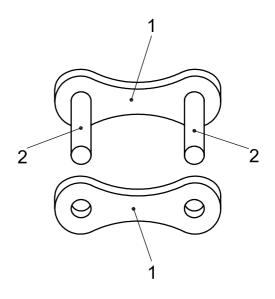
- 1 inner plate
- 2 roller
- 3 bush

a) Inner link



Key

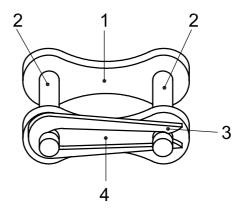
- 1 outer plate
- 2 cottered connecting pin
- 3 cotter
- 4 detachable plate
 - c) Connecting pin with cottered pin



Key

- 1 outer plate
- 2 pin

b) Outer link



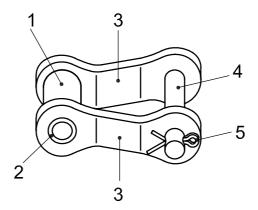
Key

- 1 outer plate
- 2 spring clip connecting pin
- 3 spring clip

Not for Resale

detachable plate

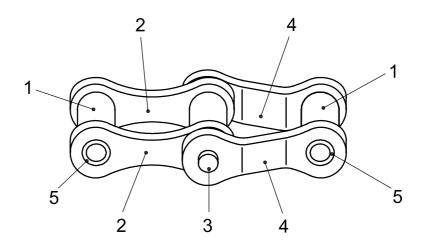
d) Spring clip connecting link



Key

- 1 roller
- 2 bush
- 3 cranked plate
- 4 cranked link pin
- 5 cotter

e) Single cranked link



Key

- 1 roller
- 2 inner plate
- 3 pin, riveted
- 4 cranked plate
- 5 bush

f) Double cranked link

- NOTE 1 Plate dimensions are shown in Table 1.
- NOTE 2 Fasteners can be of various designs. Figures give examples.

Figure 2 — Types of link

3.2 Designation

Double-pitch transmission precision roller chains are designated by the standard ISO chain numbers given in Table 1. These chain numbers have been obtained by taking the standard ISO chain number for the base chain (given in ISO 606) and adding the prefix 2.

EXAMPLE Chain ISO 1275-208B

3.3 **Dimensions**

Chains shall conform to the dimensions shown in Figure 3 and given in Table 1. Maximum and minimum dimensions are specified to ensure the interchangeability of links produced by different makers of chain. They represent limits for interchangeability, but are not the manufacturing tolerances.

In this International Standard, only double-pitch simplex chain dimensions are shown.

Tensile testing 3.4

3.4.1 General

The minimum tensile strength is that value which shall be exceeded when a tensile force is applied to a sample tested to destruction as defined in 3.4.2. This minimum tensile strength is not a working load but is intended primarily as a comparative figure between chains of various constructions.

These testing requirements do not apply to cranked links, connecting links or chains with attachments as their tensile strength could be reduced.

3.4.2 Testing

A tensile force shall be applied slowly to the ends of a chain length, containing at least five free pitches, by means of shackles permitting free movement on both sides of the chain centre-line, in the normal plane of articulation.

Failure shall be considered to have occurred at the first point where increasing extension is no longer accompanied by increasing force, i.e. the summit of the force/extension diagram. The force at this point must exceed the minimum tensile strength stated in Table 1.

Tests in which failures occur adjacent to the shackles shall be disregarded.

The tensile test shall be considered a destructive test. Even though a chain may not visibly fail when subjected to a force equivalent to the minimum tensile strength, it will have been stressed beyond the yield point and will be unfit for service.

Preloading 3.5

It is recommended that all chains be preloaded by applying a minimum tensile force equivalent to 30 % of the minimum tensile strength given in Table 1.

Table 1 — Principal chain dimensions, measuring forces and tensile strengths (see Figure 3)

| elianat muminiM Strength | | kN | 13,9 | 17,8 | 21,8 | 22,2 | 31,3 | 28,9 | 55,6 | 0,09 | 87,0 | 95,0 | 125,0 | 160,0 | 200,0 | 250,0 | |
|---|-------|----|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|--|
| Measuring force | | Z | 120 | 120 | 200 | 200 | 280 | 280 | 200 | 200 | 780 | 780 | 1 110 | 1 110 | 1 510 | 2 000 | |
| Maximum additional width for oint fastener ^c | b_7 | | 3,9 | 3,9 | 1,1 | 1,1 | 4,6 | 4,6 | 5,4 | 5,4 | 6,1 | 6,1 | 9,9 | 9'9 | 7,4 | 6,7 | |
| Maximum width over bearing pin | b_4 | | 17,8 | 17,0 | 21,8 | 19,6 | 26,9 | 22,7 | 33,5 | 36,1 | 41,1 | 43,2 | 50,8 | 53,4 | 65,1 | 67,4 | |
| Minimum width between outer plates | b_3 | | 11,31 | 11,43 | 13,97 | 13,41 | 17,88 | 15,75 | 22,74 | 25,58 | 27,59 | 29,14 | 35,59 | 38,05 | 46,71 | 45,70 | . |
| Maximum width over inner link | b_2 | | 11,17 | 11,30 | 13,84 | 13,28 | 17,75 | 15,62 | 22,60 | 25,45 | 27,45 | 29,01 | 35,45 | 37,92 | 46,58 | 45,57 | odania aieda |
| Minimum cranked link ^b | 1,1 | | 6,9 | 6,9 | 8,4 | 8,4 | 6'6 | 6'6 | 13 | 13 | 16 | 16 | 19,1 | 19,1 | 21,3 | 24,4 | end on transmission chains: add enffiv "I " to the chain number |
| Maximum plate depth | h_2 | | 12,07 | 11,81 | 15,09 | 14,73 | 18,10 | 16,13 | 24,13 | 21,08 | 30,17 | 26,42 | 36,20 | 33,40 | 37,08 | 42,29 | His ppc .su |
| Minimum chain path depth | h_1 | mm | 12,33 | 12,07 | 15,35 | 14,99 | 18,34 | 16,39 | 24,39 | 21,34 | 30,48 | 26,68 | 36,55 | 33,73 | 37,46 | 42,72 | iedo noissim |
| Minimum bush bore | d_3 | | 4,00 | 4,50 | 5,12 | 5,13 | 86'9 | 22'9 | 96'2 | 8,33 | 95'6 | 10,24 | 11,14 | 14,68 | 15,95 | 17,86 | and on trans |
| Maximum bearing pin body diameter | d_2 | | 3,98 | 4,45 | 5,09 | 2,08 | 96'9 | 5,72 | 7,94 | 8,28 | 9,54 | 10,19 | 11,11 | 14,63 | 15,90 | 17,81 | - |
| Minimum width between inner plates | b_1 | | 7,85 | 7,75 | 9,40 | 9,65 | 12,57 | 11,68 | 15,75 | 17,02 | 18,90 | 19,56 | 25,22 | 25,40 | 66'08 | 66'08 | I arra rollare are unincipally for conveyor chaine but are sometimes |
| Maximum roller diameter (large) ^a | d_7 | | 15,88 | 15,88 | 19,05 | 19,05 | 22,23 | 22,23 | 28,58 | 28,58 | 29'68 | 29'68 | 44,45 | 44,45 | I | - | iedo royova |
| Maximum roller diameter (small) ^a | d_1 | | 7,92 | 8,51 | 10,16 | 10,16 | 11,91 | 12,07 | 15,88 | 15,88 | 19,05 | 19,05 | 22,23 | 25,4 | 27,94 | 29,21 | oinally for co |
| hэйq | d | | 25,4 | 25,4 | 31,75 | 31,75 | 38,1 | 38,1 | 8'09 | 8'09 | 63,5 | 63,5 | 76,2 | 76,2 | 6'88 | 101,6 | lere are pring |
|) Chain number | osi | | 208A | 208B | 210A | 210B | 212A | 212B | 216A | 216B | 220A | 220B | 224A | 224B | 228B | 232B | a I ardo rol |

Large rollers are principally for conveyor chains but are sometimes used on transmission chains; add suffix "L" to the chain number.

b Cranked links are not recommended for use in highly stressed applications.

The actual dimensions will depend on the type of fastener used but should not exceed the dimensions given and should be obtained by the purchaser from the manufacturer.

3.6 Length validation

Measurement of chains shall take place after preloading but before lubricating.

The standard length for measurement shall be a minimum of

- a) 610 mm for ISO chain numbers 208A to 210B inclusive,
- b) 1 220 mm for ISO chain numbers 212A to 232B inclusive.

The chain shall be supported throughout its entire length and the measuring force given in Table 1 shall be applied.

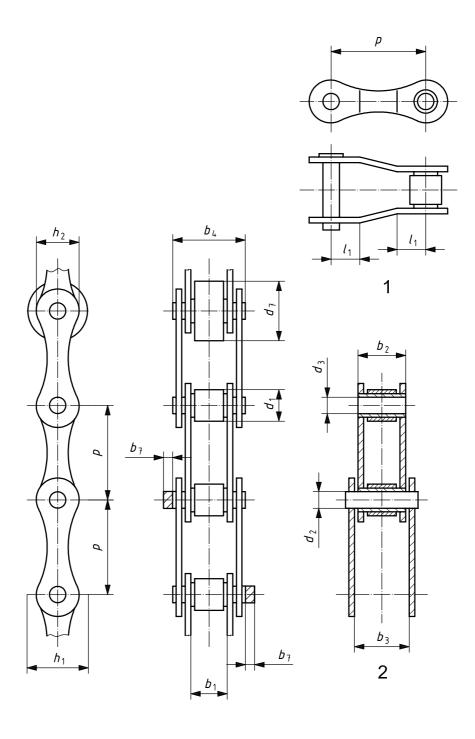
The measured length shall be the nominal length subject to tolerance of ${}^{+0,15}_{0}$ %.

The length accuracy of chains which have to work in parallel may be matched within closer tolerances.

3.7 Marking

The chains shall be marked with the manufacturer's name or trademark.

The chain number quoted in Table 1 should be marked on the chain.



Key

- 1 single cranked link
- 2 chain section

The chain path depth, h_1 , is the minimum depth of channel through which the assembled chain with small rollers will pass.

The overall width of a chain with a joint fastener is given by

 b_4 + b_7 for riveted pin end and fastener on one side,

 b_4 + 1,6 b_7 for headed pin end and fastener on one side, and

 $b_4 + 2b_7$ for fasteners on both sides.

Figure 3 — Chains

Conveyor chains

General

Except where otherwise stated, the shapes, dimensions and test details for the chain and sprockets shall conform to the requirements of Clauses 3 and 5, respectively, substituting reference to Table 2 in place of Table 1 where appropriate.

It is usual for chains used for conveyor purposes to have straight-sided (not waisted) side plates. Additionally, an alternative large roller of diameter d_7 may be adopted. These features are illustrated in Figure 4.

Nomenclature

The nomenclature of Figure 2 is also applicable to conveyor chains. Figures 2 and 4 do not define the actual form of the chain plates.



Figure 4 — Conveyor chain with large rollers

4.3 Designation

Double-pitch precision roller chains suitable for conveyor purposes are designated additionally with the prefix C when straight-sided chain plates (see Figure 4) are used, and with the suffix L when the alternative large size roller (d_7) is adopted. When necessary for the purpose of distinguishing between the chain roller sizes adopted, the optional suffix S may be applied to chains with the smaller roller.

Dimensions 4.4

When the large roller size is adopted, dimension d_1 shall be replaced by d_7 in sprocket formula. Dimensions shall be in accordance with Table 2.

Length validation

The measured length of plain chain shall be the nominal length subject to a tolerance of ${}^{+0,15}_{0}$ %.

The measured length of the chain with attachments shall be the nominal length subject to a tolerance of $^{+0,30}_{0}$ %.

The length accuracy of chains which have to work in parallel may be matched within closer tolerances.

Marking

Chains shall be marked with the manufacturer's name or trademark. The chain number quoted in Table 2 should be marked on the chain.

Table 2 — Principal conveyor chain dimensions, measuring forces and tensile strengths

| əliznət muminil trength | | κN | 13,9 | 17,8 | 21,8 | 22,2 | 31,3 | 31,3 | 58,9 | 9'55 | 9'55 | 0'09 | 0'28 | 0'28 | 0'96 | 125,0 | 125,0 | 160,0 | 222,4 | |
|--|-----------------------|----|-------|-------|-------|-------|-------|---------|-------|-------|---------|-------|-------|---------|-------|-------|---------|-------|---------|--|
| leasuring force | W | z | 120 | 120 | 200 | 200 | 280 | 280 | 280 | 200 | 200 | 200 | 780 | 780 | 780 | 1 110 | 1 110 | 1 110 | 2 000 | |
| Maximum additional width for joint tastener [©] | 29 | | 3,9 | 3,9 | 4,1 | 4,1 | 4,6 | 4,6 | 4,6 | 5,4 | 5,4 | 5,4 | 6,1 | 6,1 | 6,1 | 9,9 | 9,9 | 9,9 | 6,7 | not waisted) |
| Maximum width over bearing pin | b_4 | | 17,8 | 17,0 | 21,8 | 19,6 | 26,9 | 30,2 | 22,7 | 33,5 | 37,4 | 36,1 | 41,1 | 44,5 | 43,2 | 8'09 | 25,0 | 53,4 | 69,4 | re straight (|
| Minimum width between outer plates | p_3 | | 11,31 | 11,43 | 13,97 | 13,41 | 17,88 | 19,56 | 15,75 | 22,74 | 24,41 | 25,58 | 27,59 | 29,24 | 29,14 | 35,59 | 37,31 | 38,05 | 47,02 | side plates a |
| Maximum width over inner link | p_2 | | 11,17 | 11,30 | 13,84 | 13,28 | 17,75 | 19,43 | 15,62 | 22,60 | 24,28 | 25,45 | 27,45 | 29,10 | 29,01 | 35,45 | 37,18 | 37,92 | 46,88 | e 1, with the addition of the large roller diameters. Normally, the side plates are straight (not waisted) |
| Minimum cranked link ^b | 1,1 | | 6'9 | 6'9 | 8,4 | 8,4 | 6'6 | 6'6 | 6'6 | 13 | 13 | 13 | 16 | 16 | 16 | 19,1 | 19,1 | 19,1 | 25,2 | iameters. No |
| Maximum plate depth | h ₂ | | 12,07 | 11,81 | 15,09 | 14,73 | 18,10 | 18,10 | 16,13 | 24,13 | 24,13 | 21,08 | 30,17 | 30,17 | 26,42 | 36,20 | 36,20 | 33,40 | 48,26 | arge roller d |
| Minimum chain path depth | μ_1 | mm | 12,33 | 12,07 | 15,35 | 14,99 | 18,34 | 18,34 | 16,39 | 24,39 | 24,39 | 21,34 | 30,48 | 30,48 | 26,68 | 36,55 | 36,55 | 33,73 | 48,74 | ition of the l |
| • Minimum bush bore | q_3 | | 4,00 | 4,50 | 5,12 | 5,13 | 2,98 | 2,98 | 2,77 | 2,96 | 2,96 | 8,33 | 9;6 | 9;6 | 10,24 | 11,14 | 11,14 | 14,68 | 14,31 | with the add |
| Maximum bearing pin body diameter | d_2 | | 3,98 | 4,45 | 5,09 | 5,08 | 5,96 | 5,96 | 5,72 | 7,94 | 7,94 | 8,28 | 9,54 | 9,54 | 10,19 | 11,11 | 11,11 | 14,63 | 14,29 | |
| Minimum width between inner plates | <i>b</i> ₁ | | 7,85 | 7,75 | 9,40 | 9,65 | 12,57 | 12,57 | 11,68 | 15,75 | 15,75 | 17,02 | 18,90 | 18,90 | 19,56 | 25,22 | 25,22 | 25,40 | 31,55 | al to those |
| Maximum roller diameter (large) | d_7 | | 15,88 | 15,88 | 19,05 | 19,05 | 22,23 | 22,23 | 22,23 | 28,58 | 28,58 | 28,58 | 39,67 | 39,67 | 39,67 | 44,45 | 44,45 | 44,45 | 57,15 | s are identic |
| Maximum roller diameter (small) | a_1 | | 7,92 | 8,51 | 10,16 | 10,16 | 11,91 | 11,91 | 12,07 | 15,88 | 15,88 | 15,88 | 19,05 | 19,05 | 19,05 | 22,23 | 22,23 | 25,4 | 28,58 | n dimension |
| Pitch | р | | 25,4 | 25,4 | 31,75 | 31,75 | 38,1 | 38,1 | 38,1 | 8'09 | 8'09 | 8'09 | 63,5 | 63,5 | 63,5 | 76,2 | 76,2 | 76,2 | 101,6 | The basic chain dimensions are identical to those of Tab |
| ⁶ Tədmun nishO OS | SI . | | C208A | C208B | C210A | C210B | C212A | C212A-H | C212B | C216A | C216A-H | C216B | C220A | C220A-H | C220B | C224A | C224A-H | C224B | C232A-H | NOTE The |

The chain numbers are derived from the basic ISO chain numbers of Table 1 with the prefix C (for conveyor) and adding, as appropriate, the suffix S (for small roller) or L (for large roller). Heavy or higher-strength chains are designated with the suffix H.

b Cranked links are not recommended for use in highly stressed applications.

The actual dimensions will depend on the type of fastener used but should not exceed the dimensions given and should be obtained by the purchaser from the manufacturer.

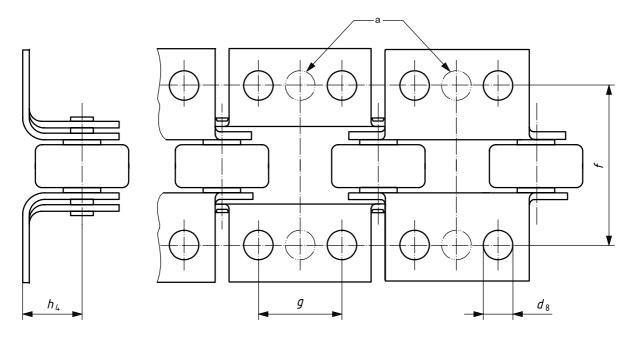
Attachments

General 4.7.1

Except where otherwise stated, the dimensions and test details for the chain with attachments shall conform to the requirements of Clause 3.

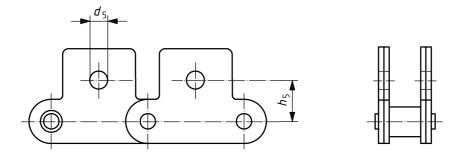
4.7.2 Nomenclature

The nomenclature for chain attachments is given in Figures 5, 6, 7, 8 and 9.



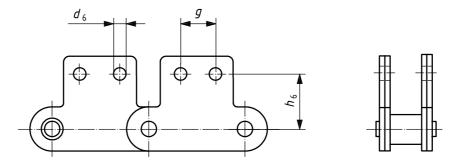
K2 attachment plates each have two attachment holes. K1 plates are similar except that they have one centrally located hole (see 4.7.3)

Figure 5 — K attachment plates



NOTE M1 attachment plates can be positioned on either inner or outer links.

Figure 6 — M1 attachment plates



NOTE M2 attachment plates can be positioned on either inner or outer links.

Figure 7 — M2 attachment plates

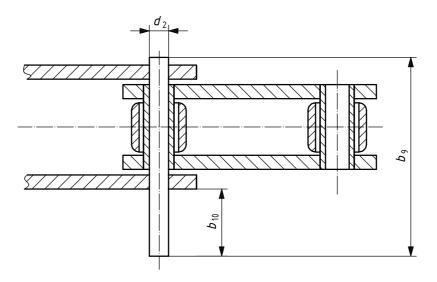


Figure 8 — Type X extended pin (figure based on duplex pin)

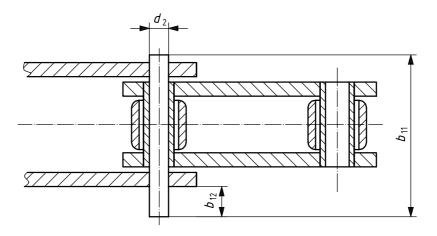


Figure 9 — Type Y extended pin (commonly used in "A" series chains)

4.7.3 Designation

Three types of attachment are given with the common dimensional basis as given in Tables 3, 4, 5 and 6. Their designation and distinguishing features are as follows.

a) K attachments (see Figure 5)

K1: with one attachment hole centrally located in each platform.

K2: with two attachment holes longitudinally located.

b) M attachments (see Figures 6 and 7)

M1: with one attachment hole centrally located in the plate.

M2: with two attachment holes longitudinally located.

c) Extended pin (see Figures 8 and 9)

with the bearing pin extended on one side of the chain.

4.7.4 Dimensions

Attachments shall conform to the dimensions given in Tables 3, 4, 5 and 6.

Table 3 — K type attachment plate dimensions (see Figure 5)

Dimensions in millimetres

| ISO chain number ^a | Platform height | Transverse distance between hole centres | Minimum hole diameter | Longitudinal distance between hole centres |
|----------------------------------|--------------------|--|--------------------------|--|
| number | h_4 | f | d_8 | g |
| C 208A | 9,1 | 25,4 | 3,3 | 9,5 |
| C 208B | 9,1 | 25,4 | 4,3 | 12,7 |
| C 210A | 11,1 | 31,8 | 5,1 | 11,9 |
| C 210B | 11,1 | 31,8 | 5,3 | 15,9 |
| C 212A | 14,7 | 42,9 | 5,1 | 14,3 |
| C 212A-H | 14,7 | 42,9 | 5,1 | 14,3 |
| C 212B | 14,7 | 38,1 | 6,4 | 19,1 |
| C 216A | 19,1 | 55,6 | 6,6 | 19,1 |
| C 216A-H | 19,1 | 55,6 | 6,6 | 19,1 |
| C 216B | 19,1 | 50,8 | 6,4 | 25,4 |
| C 220A | 23,4 | 66,6 | 8,2 | 23,8 |
| C 220A-H | 23,4 | 66,6 | 8,2 | 23,8 |
| C 220B | 23,4 | 63,5 | 8,4 | 31,8 |
| C 224A | 27,8 | 79,3 | 9,8 | 28,6 |
| C 224A-H | 27,8 | 79,3 | 9,8 | 28,6 |
| C 224B | 27,8 | 76,2 | 10,5 | 38,1 |
| C 232A-H | 36,5 | 104,7 | 13,1 | 38,1 |

Table 4 — M1 type attachment plate dimensions (see Figure 6)

Dimensions in millimetres

| ISO chain number ^a | Height from chain centreline | Minimum hole diameter |
|-------------------------------|---|-----------------------|
| 130 Chain number | h_5 | d_5 |
| C 208A | 11,1 | 5,1 |
| C 208B | 13,0 | 4,3 |
| C 210A | 14,3 | 6,6 |
| C 210B | 16,5 | 5,3 |
| C 212A | 17,5 | 8,2 |
| C 212A-H | 17,5 | 8,2 |
| C 212B | 21,0 | 6,4 |
| C 216A | 22,2 | 9,8 |
| C 216A-H | 22,2 | 9,8 |
| C 216B | 23,0 | 6,4 |
| C 220A | 28,6 | 13,1 |
| C 220A-H | 28,6 | 13,1 |
| C 220B | 30,5 | 8,4 |
| C 224A | 33,3 | 14,7 |
| C 224A-H | 33,3 | 14,7 |
| C 224B | 36,0 | 10,5 |
| C 232A-H | 44,5 | 19,5 |
| a Heavy or higher-stre | ength chains are designated with the suff | īx H. |

Table 5 — M2 attachment plate dimensions (see Figure 7)

Dimensions in millimetres

| ISO chain number ^a | Height from chain centreline | Minimum hole diameter | Longitudinal distance between hole centres |
|-------------------------------|--|-----------------------|--|
| | h_{6} | d_{6} | g |
| C 208A | 13,5 | 3,3 | 9,5 |
| C 208B | 13,7 | 4,3 | 12,7 |
| C 210A | 15,9 | 5,1 | 11,9 |
| C 210B | 16,5 | 5,3 | 15,9 |
| C 212A | 19,0 | 5,1 | 14,3 |
| C 212A-H | 19,0 | 5,1 | 14,3 |
| C 212B | 18,5 | 6,4 | 19,1 |
| C 216A | 25,4 | 6,6 | 19,1 |
| C 216A-H | 25,4 | 6,6 | 19,1 |
| C 216B | 27,4 | 6,4 | 25,4 |
| C 220A | 31,8 | 8,2 | 23,8 |
| C 220A-H | 31,8 | 8,2 | 23,8 |
| C 220B | 33,0 | 8,4 | 31,8 |
| C 224A | 37,3 | 9,8 | 28,6 |
| C 224A-H | 37,3 | 9,8 | 28,6 |
| C 224B | 42,7 | 10,5 | 38,1 |
| C 232A-H | 50,8 | 13,1 | 38,1 |
| a Heavy or higher-strengt | h chains are designated with the suffix H. | | |

Table 6 — Extended pin dimensions (see Figures 8 and 9)

Dimensions in millimetres

| | | Pin diameter | | | |
|---------------------|-----------------------------------|-----------------|-----------------------|--------------------------------|------------|
| ISO chain | Type "X' | ,, | Type " | Pili diameter | |
| number ^a | _{b₁₀} max. | $b_{ m 9}$ max. | $b_{12} \ 	ext{max}.$ | <i>b</i> ₁₁ max. | d_2 max. |
| C 208A | _ | _ | 10,2 | 26,3 | 3,98 |
| C 208B | 15,5 | 31,0 | _ | _ | 4,45 |
| C 210A | _ | _ | 12,7 | 32,6 | 5,09 |
| C 210B | 18,5 | 36,2 | _ | _ | 5,08 |
| C 212A | _ | _ | 15,2 | 40,0 | 5,96 |
| C 212A-H | _ | _ | 15,2 | 43,3 | 5,96 |
| C 212B | 21,5 | 42,2 | _ | _ | 5,72 |
| C 216A | _ | _ | 20,3 | 51,7 | 7,94 |
| C 216A-H | _ | _ | 20,3 | 55,3 | 7,94 |
| C 216B | 34,5 | 68,0 | _ | _ | 8,28 |
| C 220A | _ | _ | 25,4 | 63,8 | 9,54 |
| C 220A-H | _ | _ | 25,4 | 67,2 | 9,54 |
| C 220B | 39,4 | 79,7 | _ | _ | 10,19 |
| C 224A | _ | _ | 30,5 | 78,6 | 11,11 |
| C 224A-H | _ | _ | 30,5 | 82,4 | 11,11 |
| C 224B | 51,4 | 101,8 | _ | _ | 14,63 |
| C 232A-H | _ | _ | 40,6 | 106,3 | 14,29 |

4.7.5 Manufacture

The actual form of the attachment plates is left to the discretion of the manufacturer.

The length of the attachment plate is also left to the discretion of the manufacturer, but it should be sufficient to accommodate the two attachment holes longitudinally. In the case of attachment type K2, it should not interfere with the working of the adjoining links. A common length is normally adopted for both one and two hole attachments.

4.7.6 Marking

It is not a requirement that K and M attachment plates are marked.

The marking of the extended pin chain shall be the same as for a chain with no attachments (see 4.6).

5 Sprockets

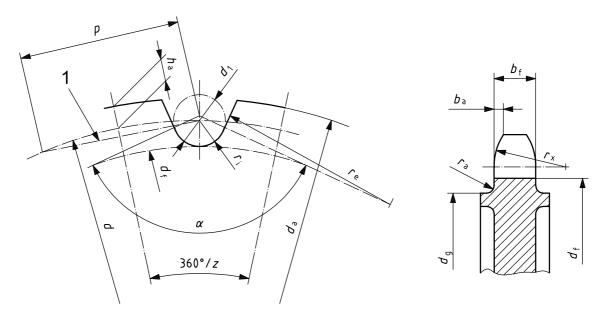
5.1 Nomenclature

The nomenclature for basic chain dimensions on which all sprocket data are based is shown in Figure 3 and given in Tables 1 and 2.

5.2 Diametral dimensions and tooth shape

5.2.1 Nomenclature

Nomenclature for diametral dimensions and tooth shape is shown in Figure 10.



Key

- 1 pitch polygon
- $b_{\rm a}$ tooth side relief
- $b_{\rm f}$ tooth width
- d pitch circle diameter
- d tip diameter
- d_f root diameter
- d_{g} absolute maximum shroud diameter
- d₁ maximum roller diameter
- $h_{\rm a}$ height of tooth above pitch polygon
- h₂ maximum plate depth

- p chordal pitch (equal to chain pitch)
- r_a shroud fillet radius
- $r_{\rm e}$ tooth flank radius
- r_i roller seating radius
- r_{\star} tooth side radius
- number of teeth corresponding to number of links that can be wrapped around the sprocket
- z_1 number of teeth for double-cut sprockets (equal to 2z)
- lpha roller seating angle

Figure 10 — Diametral dimensions and tooth shape

5.2.2 Diametral dimensions

5.2.2.1 Pitch circle diameter, d

The chain sprocket pitch circle diameter *d* is given by

$$d = \frac{p}{\sin \frac{180^{\circ}}{z}}$$

Annex A gives the pitch circle diameter for unit pitch as a function of the number of teeth.

5.2.2.2 Measuring pin diameter, d_R

The chain sprocket measuring pin diameter, d_R , is given by

$$d_{\rm R} = d_{1 \ 0}^{+0.01}$$
 (see Figure 11)

5.2.2.3 Root diameter, d_f

The chain sprocket root diameter d_f is given by

$$d_{\mathsf{f}} = d - d_{\mathsf{1}}$$

and is subject to the tolerance limits given in Table 7.

Table 7 — Chain sprocket root diameter tolerance limits

Dimensions in millimetres

| Root diameter $d_{\rm f}$ | Upper deviation | Lower deviation |
|-----------------------------|-----------------|-----------------|
| <i>d</i> _f ≤ 127 | 0 | 0,25 |
| $127 < d_{f} \leqslant 250$ | 0 | 0,3 |
| d _f > 250 | h1 | 1 ^a |
| a See ISO 286-2. | | |

5.2.2.4 Measurement over pins (see Figure 11)

For an even number of teeth, the measurement over pins is given by

$$M_{R, min} = d + d_{R}$$

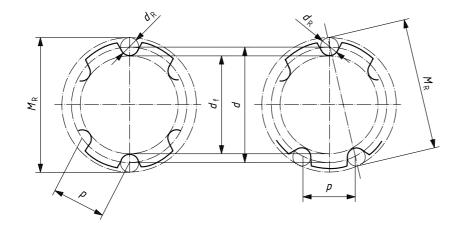
For an odd number of teeth and single-cut sprockets, the measurement over pins is given by

$$M_{\rm R, min} = d \cos \frac{90^{\circ}}{z} + d_{\rm R}$$

Not for Resale

For an odd number of teeth and double-cut sprockets, the measurement over pins is given by

$$M_{\rm R, min} = d\cos\frac{90^{\circ}}{z_1} + d_{\rm R}$$



Even number of teeth

Odd number of teeth

Single- and double-cut teeth

Full line = zDash-dotted line = 2z

Key

d pitch circle diameter

d_f root diameter

d_R measuring pin diameter

 M_{R} measurement over pins

p chordal pitch, equal to chain pitch

Figure 11 — Measuring pin size

The measurement over pins of sprockets with an even number of teeth shall be carried out over pins inserted in opposite tooth gaps.

The measurement over pins of sprockets with an odd number of teeth shall be carried out over pins in the tooth gaps most nearly opposite.

The limits of tolerance for measurement over pins are identical to those for the corresponding root diameters.

NOTE 1 Sprockets for double-pitch chains can be provided with either one set of teeth having a number of teeth equal to z or with two sets of teeth, the location of the tooth spaces of the second set being midway between those of the first. In this case, the total number of teeth of the sprockets will be $z_1 = 2z$.

NOTE 2 In the case of a single set of teeth z will be integer. In the case of double-cut sprockets z_1 will be an integer, but z will be fractional if z_1 is an odd number.

Short-pitch base chains should not be used with double-cut sprockets made for double-pitch chains or vice-versa.

5.2.2.5 Tip diameter

The maximum and minimum diameters of the chain sprocket tip are given by

$$d_{a. max} = d + 0,625p - d_1$$

$$d_{a, min} = d + p \left(0.5 - \frac{0.4}{z} \right) - d_1$$

It should be noted that $d_{\rm a,\ max}$ and $d_{\rm a,\ min}$ can be applied arbitrarily both to the maximum and minimum tooth gap forms, subject to the limitations imposed by the cutter on the maximum diameter.

To facilitate the construction of the tooth gap form to a large scale, the tooth height above the pitch polygon can be obtained from the following formula:

$$h_{a, \text{max}} = p \left(0.3125 + \frac{0.8}{z} \right) - 0.5d_1$$

$$h_{\text{a, min}} = p \left(0.25 + \frac{0.6}{z} \right) - 0.5d_1$$

NOTE

 $h_{a, \text{max}}$ is related to $d_{a, \text{max}}$ and $h_{a, \text{min}}$ to $d_{a, \text{min}}$.

Sprocket tooth gap form

5.2.3.1 General

The actual tooth gap form which is provided by cutting or by an equivalent method shall have tooth flanks of a form lying within the minimum and maximum flank radii and blending smoothly with the roller seating curve subtending the respective angles.

5.2.3.2 Minimum form

The corresponding values for r_e , r_i and α are given by

$$r_{\rm e, \, min} = 0.12 \, d_1 \, (z + 2)$$

$$r_{\rm i.\ min} = 0,505\ d_1$$

$$\alpha_{\text{max}} = 140^{\circ} - \frac{90^{\circ}}{7}$$

5.2.3.3 **Maximum form**

The corresponding values for $r_{\rm e}$, $r_{\rm i}$ and α are given by

$$r_{\rm e.\ max} = 0.008\ d_1\ (z^2 + 180)$$

$$r_{i,\text{max}} = 0.505d_1 + 0.069 \sqrt[3]{d_1}$$

$$\alpha_{\min} = 120^{\circ} - \frac{90^{\circ}}{z}$$

5.2.3.4 Tooth width

The sprocket tooth width b_f is given by

$$b_{\rm f} = 0.95 \ b_{\rm 1}$$
: h14¹⁾

 $b_f = 0.93 \ b_1$: h14 may be used by agreement between user and manufacturer.

5.2.3.5 Tooth side relief

The sprocket tooth side relief $b_{a, nom}$ is given by

$$b_{a, nom} = 0.065p$$

5.2.3.6 Absolute maximum shroud diameter

The sprocket absolute maximum shroud diameter $d_{\mathbf{q}}$ is given by

$$d_{\rm g} = p \cot \frac{180^{\circ}}{z} - 1,05 h_2 - 1 - 2r_{\rm a}$$

5.2.3.7 Tooth side radius

The sprocket tooth side radius $r_{x, nom}$ is given by

$$r_{x. nom} = 0.5p$$

5.3 Radial run-out

Radial run-out, measured on one complete turn of the sprocket, with reference to the bore and root diameter, shall not exceed a total indicator reading greater than the larger of the two following values:

$$(0,000 \ 8 \ d_f + 0,08) \ \text{mm} \ \text{or} \ 0,15 \ \text{mm}$$

up to a maximum of 0,76 mm.

5.4 Axial run-out (wobble)

Axial run-out, measured on one complete turn of the sprocket with reference to the bore and the flat part of the side face of the teeth, shall not exceed a total indicator reading of

$$(0,000 \ 9 \ d_f + 0,08) \ mm$$

up to a maximum of 1,14 mm.

For fabricated (welded) sprockets, a minimum of 0,25 mm may be accepted if the above formula gives smaller values.

5.5 Pitch accuracy of sprocket teeth

Pitch accuracy of sprocket teeth is important and chain manufacturers should be consulted for details.

¹⁾ See ISO 286-2.

Bore tolerance

Unless otherwise agreed between the manufacturer and the purchaser, bore tolerances shall be H82).

5.7 Marking

Sprockets shall be marked with the following information:

- manufacturer's name or trademark; a)
- number of teeth; b)
- chain designation (ISO chain number and/or manufacturer's equivalent). c)

²⁾ See ISO 286-2.

Annex A (normative)

Pitch circle diameters

Table A.1 gives the correct pitch circle diameters for sprockets to suit a chain of unit pitch. The pitch circle diameters for sprockets to suit a chain of any other pitch are directly proportional to the pitch of the chain.

Table A.1 — Pitch circle diameters

| Number of teeth | Pitch circle diameter d for | Number of teeth | Pitch circle diameter d for | Number of teeth | Pitch circle diameter d for | Number of teeth | Pitch circle diameter d for |
|-----------------|---|-----------------|-------------------------------|-----------------|-----------------------------|-----------------|-------------------------------|
| z | unit pitch ^a | z | unit pitch ^a | z | unit pitch ^a | z | unit pitch a |
| 5 | 1,701 3 | 23 | 7,343 9 | 41 | 13,063 5 | 59 | 18,789 2 |
| 5 1/2 | 1,849 6 | 23 1/2 | 7,502 6 | 41 1/2 | 13,222 5 | 59 1/2 | 18,948 2 |
| 6 | 2 | 24 | 7,661 3 | 42 | 13,381 5 | 60 | 19,107 3 |
| 6 1/2 | 2,151 9 | 24 1/2 | 7,82 | 42 1/2 | 13,540 5 | 60 1/2 | 19,266 5 |
| 02 | 2, | | .,0= | | .0,0.00 | 002 | .0,200 |
| 7 | 2,304 8 | 25 | 7,978 7 | 43 | 13,699 5 | 61 | 19,425 5 |
| 7 1/2 | 2,458 6 | 25 1/2 | 8,137 5 | 43 1/2 | 13,858 5 | 61 1/2 | 19,584 7 |
| 8 | 2,613 1 | 26 | 8,296 2 | 44 | 14,017 6 | 62 | 19,743 7 |
| 8 1/2 | 2,768 2 | 26 1/2 | 8,455 | 44 1/2 | 14,176 5 | 62 1/2 | 19,902 9 |
| 9 | 2,923 8 | 27 | 8,613 8 | 45 | 14,335 6 | 63 | 20,061 9 |
| 9 1/2 | 3,079 8 | 27 1/2 | 8,772 6 | 45 1/2 | 14,494 6 | 63 1/2 | 20,221 |
| 10 | 3,236 1 | 28 | 8,931 4 | 46 | 14,653 7 | 64 | 20,38 |
| 10 1/2 | 3,392 7 | 28 1/2 | 9,090 2 | 46 1/2 | 14,812 7 | 64 1/2 | 20,539 3 |
| 10 1/2 | 0,002 7 | 20 1/2 | 0,000 2 | 40 1/2 | 14,012 7 | 04 1/2 | 20,000 0 |
| 11 | 3,549 4 | 29 | 9,249 1 | 47 | 14,971 7 | 65 | 20,698 2 |
| 11 1/2 | 3,706 5 | 29 1/2 | 9,408 0 | 47 1/2 | 15,130 8 | 65 1/2 | 20,857 5 |
| 12 | 3,863 7 | 30 | 9,566 8 | 48 | 15,289 8 | 66 | 21,016 4 |
| 12 1/2 | 4,021 1 | 30 1/2 | 9,725 6 | 48 1/2 | 15,448 8 | 66 1/2 | 21,175 7 |
| 13 | 4,178 6 | 31 | 9,884 5 | 49 | 15,607 9 | 67 | 21,334 6 |
| 13 1/2 | 4,336 2 | 31 1/2 | 10,043 4 | 49 1/2 | 15,766 9 | 67 1/2 | 21,493 9 |
| 14 | 4,494 | 32 | 10,202 3 | 50 | 15,700 9 | 68 | 21,652 8 |
| 14 1/2 | 4,651 8 | 32 1/2 | 10,361 2 | 50 1/2 | 16,085 | 68 1/2 | 21,812 1 |
| | ,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,, | | , | | 12,000 | | |
| 15 | 4,809 7 | 33 | 10,520 1 | 51 | 16,244 1 | 69 | 21,971 |
| 15 1/2 | 4,967 7 | 33 1/2 | 10,679 | 51 1/2 | 16,403 1 | 69 1/2 | 22,130 3 |
| 16 | 5,125 8 | 34 | 10,838 | 52 | 16,562 2 | 70 | 22,289 2 |
| 16 1/2 | 5,284 | 34 1/2 | 10,996 9 | 52 1/2 | 16,721 2 | 70 1/2 | 22,448 5 |
| 17 | 5,442 2 | 35 | 11,155 8 | 53 | 16,880 3 | 71 | 22,607 4 |
| 17 1/2 | 5,600 5 | 35 1/2 | 11,314 8 | 53 1/2 | 17,039 3 | 71 1/2 | 22,766 7 |
| 18 | 5,758 8 | 36 | 11,473 7 | 54 | 17,198 4 | 72 | 22,925 6 |
| 18 1/2 | 5,917 1 | 36 1/2 | 11,632 7 | 54 1/2 | 17,357 5 | 72 1/2 | 23,084 9 |
| 19 | 6,075 5 | 37 | 11,791 6 | 55 | 17,516 6 | 73 | 23,243 8 |
| 19 1/2 | 6,234 | 37 1/2 | 11,950 6 | 55 1/2 | 17,675 6 | 73 73 1/2 | 23,403 1 |
| 20 | 6,392 5 | 37 1/2 | 12,109 6 | 56 | 17,834 7 | 73 1/2 | 23,562 |
| 20 1/2 | 6,550 9 | 38 1/2 | 12,109 6 | 56 1/2 | 17,834 7 | 74 74 1/2 | 23,721 3 |
| 20 1/2 | 0,000 8 | JU 1/2 | 12,200 0 | JU 1/2 | 17,000 0 | 17 1/2 | 20,7210 |
| 21 | 6,709 5 | 39 | 12,427 5 | 57 | 18,152 9 | 75 | 23,880 2 |
| 21 1/2 | 6,868 1 | 39 1/2 | 12,586 5 | 57 1/2 | 18,311 9 | - | _ |
| 22 | 7,026 6 | 40 | 12,745 5 | 58 | 18,471 | - | _ |
| 22 1/2 | 7,185 3 | 40 1/2 | 12,904 5 | 58 1/2 | 18,630 1 | _ | _ |
| a This is so | ometimes referred to | as "unit pitch | circle diameter". | | | | |

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