Incorporating Amendment No. 1

Specification for

Pinion type cutters for spur gears — 1 to 8 metric module

UDC 621.914.6:621.833



Co-operating organizations

The Mechanical Engineering Industry Standards Committee, under whose supervision this British Standard was prepared, consists of representatives from the following Government departments and professional and industrial organizations:

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Manufacturers Ltd.

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Milling Cutter and Reamers Association
National Federation of Engineers Tool
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Navy and Vickers Gearing Research Association

North East Coast Institution of Engineering and Shipbuilders

This British Standard, having been approved by the Mechanical Industry Standards Committee, was published under the authority of the Executive Board on 27 February 1976

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Contents

		Page
Co-	operating organizations	Inside front cover
For	reword	ii
1	Scope	1
2	References	1
3	Terminology	1
4	Tooth form	1
5	Dimensions	1
6	Bore	2
7	Provision of datum cylinders	2
8	Tolerances	2
9	Marking	2
10	Instructions for ordering	2
App	pendix A Incorrect gear action	12
Fig	ure 1 — Cutter terms	1
Fig	ure 2 — Disc gear cutter	2
Fig	ure 3 — Extended hub gear cutter	5
Fig	ure 4 — Screwed hub gear cutter	6
Fig	ure 5(a) — Parallel shank gear cutter	6
Fig	ure 5(b) — Large stub taper shank gear cutter	6
Fig	ure 5(c) — No. 2 Morse stub taper shank gear cutter	7
Tak	ole 1(a) — Disc gear cutter dimensions (31.75 mm	
and	l 44.45 mm diameter bore)	3
	ole 1(b) — Disc gear cutter dimensions (58.738 mm diamet	er
	e, 10 mm nominal size, with optional keyway)	4
	ple 2 — Extended hub gear cutter dimensions (31.75 mm	_
	d 44.45 mm diameter bore)	5
	ble 3 — Screwed hub gear cutter dimensions	6
	ble 4 — Dimensions of parallel and large stub and	-
	2 Morse stub taper shank gear cutters	7
	ble 5 — Dimensions of bore	7
	ble 6 — Accuracy tests	8
Puk	olications referred to	Inside back cover

[↑] PCI 12-1999

Foreword

This British Standard has been prepared under the authority of the Mechanical Engineering Industry Standards Committee and is one of a number dealing with gear cutting tools.

This edition is a revision of the 1957 edition of BS 2887 but in accordance with the UK policy of metrication, diametral pitch cutters have been replaced by metric module cutters. BS 2887 is therefore withdrawn.

It should be noted that, due to established world wide usage, nominal bore sizes in inch units of $1^{1}/_{4}$, $1^{3}/_{4}$ and $2^{5}/_{16}$ have been directly converted into millimetres.

A British Standard does not purport to include all the necessary provisions of a contract. Users of British Standards are responsible for their correct application.

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Summary of pages

This document comprises a front cover, an inside front cover, pages i and ii, pages 1 to 12, an inside back cover and a back cover.

This standard has been updated (see copyright date) and may have had amendments incorporated. This will be indicated in the amendment table on the inside front cover.

1 Scope

This British Standard specifies certain dimensions and accuracy requirements for non-topping pinion type finishing cutters for generating involute spur gear teeth of 1 to 8 metric module.

The following types of cutter are covered by this standard.

Disc [see Table 1(a) and Table 1(b)].

Extended hub (see Table 2).

Screwed hub (see Table 3).

Parallel shank (see Table 4).

Large stub taper shank (see Table 4).

No. 2 Morse stub taper shank (see Table 4).

These cutters are generally intended for the production of metric module gears which comply with the requirements of ISO 53 and ISO 54 and BS 436-2. This standard, BS 5246, includes two additional modules, 3.25 and 3.75.

Two tolerance grades are given.

Grade AA. Cutters for the production of gears to fine tolerances.

Grade A. Cutters for the production of gears for general purposes.

NOTE Grade AA is applicable to all the cutters illustrated except the screwed hub type shown in Figure 5.

Details of the markings recommended for pinion type cutters and information to be given when ordering standard cutters are included.

2 References

The titles of the British Standard and International Standards referred to in this standard are listed on the inside back cover.

3 Terminology

3.1

normal module, m_n

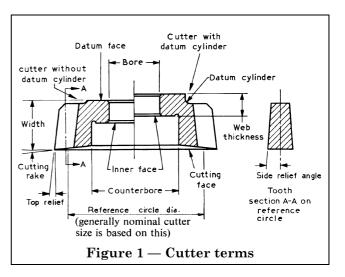
the quotient of the normal pitch, expressed in mm, by the number π

3.2 General.

The general terms used, common to gears and gear forming tools, are taken from BS 2519¹⁾.

Notations are as given in BS 2519¹⁾.

Figure 1 illustrates the terminology applied to a typical disc type cutter.



4 Tooth form

Pinion type cutters shall have teeth of such a form that they will be capable of generating gear teeth corresponding to the British Standard basic rack tooth profile shown in BS 436-2 except that slight changes in profile, root fillet and root clearance of the gear teeth will occur during the life of the cutter.

The actual dimensions of the following elements of the cutter shall be so related that when cutting a basic rack to the standard tooth thickness, the standard root clearance will be maintained as a minimum over the first 75 % of the useful face width of the cutter teeth:

- a) outside diameter,
- b) tooth thickness,
- c) side relief angle,
- d) top relief angle.

NOTE With certain combinations of gear and cutter sizes, gears produced by this process will give rise to incorrect tooth action. See Appendix A.

5 Dimensions

Dimensions for the cutters listed shall comply with those given in Table 1 to Table 4 as appropriate. Cutters are also manufactured and used outside the range of pitches indicated for each type but these are not covered by this standard. However, by agreement between the purchaser and manufacturer, essential features of this standard, e.g. tolerances, may be applied to such other designs.

For bore dimensions see clause 6.

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¹⁾ Under revision. Will also embody notations given in ISO/R 701.

6 Bore

Each end of the bore shall be broken with a small chamfer or radius not exceeding 1 mm and the bore shall be finished parallel. The dimensions of the bore shall comply with those given in Table 5.

7 Provision of datum cylinders

7.1 For grade AA cutters. All grade AA cutters shall have a datum cylinder as shown in Figure 1, Figure 2, Figure 3 and Figure 5.

7.2 For grade A disc cutters. A datum cylinder as in 7.1 above may be incorporated by arrangement between the purchaser and the manufacturer.

Other grade A cutters do not normally have a datum cylinder.

8 Tolerances

The tolerances appropriate to each type and grade of pinion cutter are given in Table 6.

In general, the tolerances for grade AA cutters apply only to the types referred to in Table 1(a), Table 1(b), Table 2 and Table 4.

9 Marking

Recommendations for the information to be marked on pinion type gear cutters are given below.

Normal module $xm_{\rm n}$

Normal pressure angle at

reference cylinder α_n Number of teeth xzGrade of accuracy of cutter ...
Cutting depth x CD

Number and year of this British

Standard BS 5246:1975

Maker's identification mark or number, and cutter material may also be added if desired.

10 Instructions for ordering

Information required by the suppliers of pinion cutters, which should be stated when ordering, is tabulated below.

Quantity required.

Type of cutter.

Normal module.

Grade of accuracy of cutter.

Nominal cutter size.

Diameter of bore or size and type of shank.

Number of this British Standard.

Example

One: Pinion type disc cutter, 3 metric module, Grade A, 100 mm nominal size, 31.75 mm bore, BS 5246.

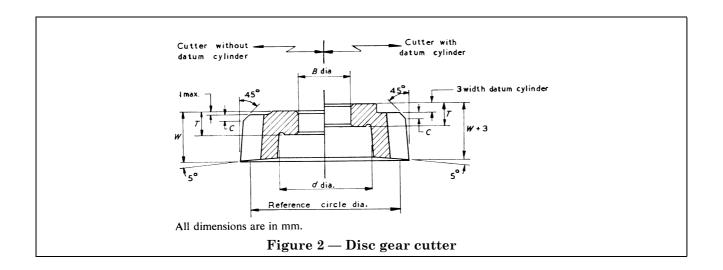


Table 1(a) — Disc gear cutter dimensions (see Figure 2)

(31.75 mm and 44.45 mm diameter bore)

All dimensions are in mm.

1	2	3	4	5	6	7
Module	75 mm nomin	nal cutter size	100 mm nomi	nal cutter size	Web thickness	Chamfer
$m_{ m n}$	Number of teeth z	Width W (without datum cylinder) (See note 1)	Number of teeth z	Width W (without datum cylinder) (See note 1)	T	C
1	76	16	100	16		
1.125	68	16	88	16		
1.25	61	16	80	16	8	1.5
1.375	56	17.5	72	19	0	1.0
1.5	51	17.5	66	19		
1.75	43	19	58	19		
2	38	22	50	22		
2.25	34	22	44	22		
2.5	30	22	40	22		
2.75	28	22	36	22		
3	25	22	33	22	10	
3.25	23	25	31	25		
3.5	22	25	29	25		3
3.75	20	25	27	25		
4	19	25	25	25		
4.5	17	25	22	25		
5			20	25		
5.5			18	29		
6			17	29		
6.5			16	29	12	
7			15	29		
8			13	29		
$\begin{array}{c} \textbf{Diameter of} \\ \textbf{bore } B \end{array}$						
	31.75		31.75 or 44.45			
Counterbore diameter d min. (See note 2)						
	53		67			

See also Appendix A.

NOTE 1 When a datum cylinder is provided this dimension will need to be increased by 3 mm. NOTE 2 The counterbore may be tapered or parallel.

3 ∩ ¤SI 12-1999

Table 1(b) — Disc gear cutter dimensions (see Figure 2)

(58.738 mm diameter bore, 100 mm nominal size, with optional keyway)

All dimensions are in mm.

1	2	3	4	5	6	7
$\begin{array}{c} \textbf{Module} \\ m_{\mathbf{n}} \end{array}$	No. of teeth	Width W (without datum cylinder) (See note 1)	Web thickness	Counterbore diameter d min. (See note 2)	Size of optional keyway in bore	Chamfer C
1	100					
1.125	88					
1.25	80					1.5
1.375	72					1.0
1.50	66					
1.75	58	25				
2	50					
2.25	44					
2.5	40		13			
2.75	36					
3	33					
3.25	31			78 mm min.	9.6 wide × 3.2 deep	
3.5	29					
3.75	27					
4	25	29				3.0
4.5	23					
5	21			1		
5.5	19					
6	18	30.5	1			
6.5	17	30.5	15			
7	16	32				
8	14	33.5				

See also Appendix A.

NOTE 1 When a datum cylinder is provided this dimension will need to be increased by 3 mm. NOTE 2 The counterbore may be tapered or parallel.

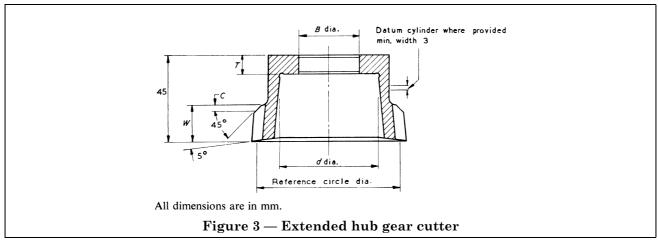


Table 2 — Extended hub gear cutter dimensions (see Figure 3)

(31.75 mm and 44.45 mm diameter bore)

All dimensions are in mm.

1	2	3	4	5	6	7
Module	75 mm nomir	al cutter size	100 mm nomi	nal cutter size	Web thickness	Chamfer
$m_{ m n}$	$\begin{array}{c} \text{Number of} \\ \text{teeth} \\ z \end{array}$	$\begin{matrix} \textbf{Width} \\ W \end{matrix}$	$\begin{array}{c} \text{Number of} \\ \text{teeth} \\ z \end{array}$	Width W	T	C
1	76	16	100	16		
1.125	68	16	88	16		
1.25	61	16	80	16		
1.375	56	17.5	72	19	9	1.5
1.5	51	17.5	66	19		
1.75	43	19	68	19		
2	38	22	50	22		
2.25	34	22	44	22		
2.5	30	22	40	22		
2.75	28	22	36	22		
3	25	22	33	22	11	
3.25	23	25	31	25		
3.5	22	25	29	25		3
3.75	20	25	27	25		
4	19	25	25	25		
4.5	17	$\frac{1}{25}$	$\frac{1}{22}$	$\frac{1}{25}$		
5			20	25		
5.5			18	29		
6			17	29		
6.5			16	29	13	
7			15	29		
8			13	29		
$\begin{array}{c} \textbf{Diameter of} \\ \textbf{bore } B \end{array}$	31.75	I	31.75 or 44.45			
Counterbore diameter d min. (See note)	53		67			
See also Appendix A	1		L		1	

See also Appendix A.

NOTE The counterbore may be tapered or parallel.

5 ∩ PCI 12-1999

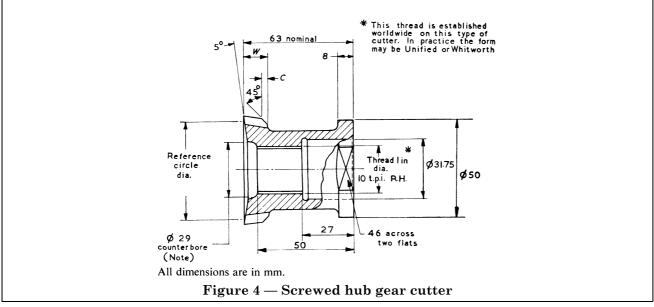
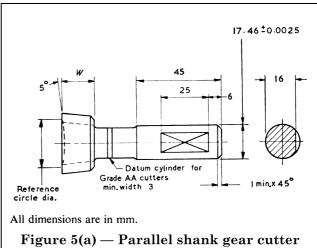


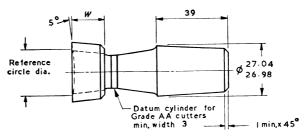
Table 3 — Screwed hub gear cutter dimensions (see Figure 4)

All dimensions are in mm.

1	2	3	4	5
$\begin{array}{c} \textbf{Module} \\ m_{\mathbf{n}} \end{array}$	No. of teeth	Reference circle diameter	Width W	Chamfer C
1	50	50	16	
1.125	45	50.625	16	1.5
1.25	40	50	16	
1.375	38	52.25	19	
1.50	35	52.5	19	
1.75	30	52.5	19	
2	28	56	22	,
2.25	25	56.25	22	
2.50	25	62.5	22	3
2.75	23	63.25	22	
3	21	63	22	
NOTE The	counterbore	may he taner	ed or paralle	1

NOTE The counterbore may be tapered or parallel.





All dimensions are in mm.

Taper of shank = 0.6255 : 12

If a taper of 0.623 26: 12 (equivalent to the no.4 Morse rate of taper) is desired, this should be stated when making

Figure 5(b) — Large stub taper shank gear cutter

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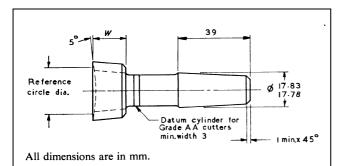


Figure 5(c) — No. 2 Morse stub taper shank gear cutter

Table 4 — Dimensions of parallel and large stub and No. 2 Morse stub taper shank gear cutters (see Figure 5)

All dimensions are in mm.

1	2	3	4
$\begin{array}{c} \textbf{Module} \\ m_{\mathbf{n}} \end{array}$	Reference circle diameter	Width W min.	$\begin{array}{c} \text{Number of} \\ \text{teeth} \\ z \end{array}$
1	25		25
1.125	22.5		20
1.25	20	8	16
1.375	24.75		18
1.50	30		20
1.75	28		16
2	28		14
2.25	29.25		13
2.50	30	10	12
2.75	33		12
3	36		12

Table 5 — Dimensions of bore

All dimensions are in mm.

1	2	3
Nominal diameter	Limits Tolera	of size nce H4
diameter	min.	max.
31.75	31.750	31.757
44.45	44.450	44.457
58.738	58.738	58.746

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Table 6 — Accuracy tests

Tolerance unit 0.001 mm unless otherwise stated.

10			8 ≥ 9 <													
6	ance	lule	$> 3.5 \leqslant 6$											0° 25°		0
œ	Tolerance	Module	$>2\leqslant3.5$	C		3	5	3	5	ιO	∞	3	ក	+ 0° - 0.25°	8	10
7			$\geqslant 1 \leqslant 2$													
9	Grade of	cutter		<u> </u>	A	AA	A	AA	A	AA	A	AA	A	AA and A	AA and A	AA and A
ro	Nominal	cutter size		mm 11x +0.7E		100		Up to 75		100		All sizes		All sizes	Up to 75	100
4	Test			Tlothood of	datum face	ly)		Axial run-out				Parallelism of	inner face with datum face	Accuracy of cutting rake angle (5° nominal)	Axial run-out	
က	Element			Detum food				Datum face	(See note 2)				e 2)	Cutting face	Cutting face	
67	Typical test arrangement	(See notes 4 and 7)		(<u> </u>						The dial indicator	must travel along radial path	The dial indicator must travel along radial path		
1	Test			5	3			G2				G3		G4	G5	

Table 6 — Accuracy tests

Tolerance unit 0.001 mm unless otherwise stated.

_	1 1					. 1						
22					> 80	$\int_{\gamma}^{2} \infty$	113	9	١	9	4	9
21			$8 \geqslant 9$								1	
20			٨			1						
19							1					
18			•		> 80	$\int_{\gamma}^{2} \infty$	133	9	٠	ဖ	4	ro
17			$3.5\leqslant 6$		> 50	8	113	9	0	9	4	ro
16			۸	ter	1	1						
15	ce	е		ame		1	1]
14	Tolerance	Module		Reference diameter	> 80		113	હ	٠	9	4	ro
13	T		$\leqslant 3.5$	efere	> 50		13	9	٠	9	4	ro
12			> 2	F	> 32	8	113	יכ	0 1	က	4	5
11					$\leqslant 32$	8	113	بر	0	က	4	5
10					> 80	8	113	6	۰ ۱	9	3	4
6			$\leqslant 2$		> 50			9	,	9	3	4
8			> 1		> 32	$\frac{2}{\sqrt{2}}$ ∞	113	יכ	0 1	က	3	4
7					≤ 32	∞	113	יכ	0 1	က	3	4
	e of	er										
9	Grade of	cutter				AA	A	AA	* 7* 7	A	AA	A
	nal	er	מ			sez		202	2		sez	
5	Nominal	cutter	212			All sizes		All sizes			All sizes	
										9	1	
4	Test					out,	ive i	1110-	, , ,	31ve 1	ile	<u>.</u>
						Run-out,	relative to	Rim-out	11001	relative to bore	Profile	error
	int					4			c	r of		tes 1 6)
အ	Element					Outside	diameter	Datum	117	cylinder of disc type cutter	th	(see notes 3, 5 and 6)
	H					On	dia	Dat	7 ;	cylind disc ty cutter	Tooth	(Se 3), E
	Typical test arrangement	3 4 and 7)				6		7			\ \hat{\state}	Diameter corresponding to mid-cutting depth
2	pical test a	(See notes 4 and 7)						Datum cylinder				Cutting depth Design profile is represented by full line
								Datu				Designation of the particular
П	Test					99	5	G7	5		G8	

Table 6 — Accuracy tests

Tolerance unit 0.001 mm unless otherwise stated.

	2		4	ro	9	7	∞	6	10	111	12	13	14	15	16 1	17	18	19 20	0 21	22	01
Test	Typical test arrangement	Element	Test	Nominal	Grade of							To	Tolerance	3e							
	(See notes 4 and 7)			cutter	cutter							N	Module	0							
							×	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8			2 × 3 ×	≤ 3.5			> 3.5	9 >>			> 9 <	8	
											R	Reference diameter	ice dia	amete	ı						
						≥ 32	> 32 ≤ 50	> 50 	> 80 < 125	≥ 32	> 32 < 50	> 50 80	> 80 < 125	İ	^ \/	50 80	> 80 -			> 80 125	80 125
_		Tooth	Run-out of	All sizes AA	AA	∞	6	6	6	6	6	6	6		- 10	01 10	-			12	
	reading measured with a ball ended plunger contacting near the reference circle in each tooth space		reference circle,		A	13	13	114	14	14	14	. 15	15		16	3 16		1	1	19	
	Adjacent	Tooth	Adjacent pitch.	All sizes AA	AA	က	33	3	3	3	က	33	3		1	4	4			4	
	pitch 2		(To be measured on each set of flanks independently)		A	4	4	4	4	ю	ರ	ro	ರ			9	9			L	

Table 6—Accuracy tests

stated.
erwise
othe
unless
lun mm
\Box
90
$\ddot{\circ}$
unit
Tolerance

=								
	22					> 80 < 125	6	14
	21			8				
	20			3 ≫ 9 <			İ	
	19						1	1
	18					> 80 < 125	6	14
	11			> 3.5 \le 6		> 50 < 80	<i>L</i>	11
	16	Tolerance	Module	۸	Reference diameter			
	15						1	
	14					> 80 < 125	6	14
	13	T		$2\leqslant 3.5$	efere	> 50 < 80	<i>L</i>	11
	12			^	R	> 32 < 50	9	10
	11			$\geqslant 1 \leqslant 2$		< 32	5	∞
	10					> 80 < 125	6	14
	6					> 50 8 80	7	11
	8			$ \rangle$		> 32 < 50	9	10
	7					≤ 32	5	∞
	9	Grade of cutter					AA	A
	5	Nominal cutter size					All sizes AA	
	4	Test						pitch over an arc of the reference circle not exceeding 180°
	3	Element					Tooth	
	2	Typical test arrangement (See notes 4 and 7)					Approx.	length of arc of measurement measurement
	1	Test					G11	

Unless otherwise stated the values represent the full indicator movement (FIM). NOTE 1 NOTE 2 NOTE 3 NOTE 4

Profile tolerance shall be related to "unmodified" portion of basic rack. Applies to disc and extended hub cutters only.

On Grade A cutters other than the disc type, the portion of the hub or shank adjacent to the cutting portion is the datum cylinder to which the tolerances are related. When the profile is measured at the cutting face, the pressure angle has the standard value of 20°. (For this type of measurement a stylus with a flat anvil will need to be used).

 $\begin{pmatrix} \tan 20^{\circ} \\ 1-\tan 5^{\circ} \tan \alpha_{c} \end{pmatrix} \text{where } \alpha_{c} \text{ is the top}$ NOTE 6 When the profile is measured behind the cutting face in a transverse plane the pressure angle varies from the standard value and is equal to tan⁻¹

relief angle of the cutter.

NOTE 7 For convenience the drawings show only one type of cutter, but where appropriate, these tests also apply to screwed hub gear cutters, extended hub cutters and shank type cutters.

Appendix A Incorrect gear action

There are two main geometrical reasons which are likely to cause incorrect gear meshing action; these are described below.

- a) If a cutter is used which is much smaller than the gear mating to the one being cut, there may be some interference between the root fillet generated by the cutter and the tip diameter of the mating gear. Thus it is necessary to consider this when choosing the size and type of cutter to be used. This condition becomes dangerous when cutting pinions where the mating wheel is large, worse when the mating gear is a rack, and much worse when the mating gear is an internal gear.
- b) If a cutter with a very low number of teeth is used, involute interference will occur in the form of excessive tip modification on the gear. When cutting external gears this condition becomes worse on large gears; when cutting internal gears the condition becomes worse on small gears. In the latter case, while it is necessary to use a larger cutter to avoid or alleviate this condition, it must also be remembered that cutters for internal gears have a limit to their maximum size to avoid an interference condition which can occur during infeed.

Generally the cutter user is not in a position to analyse any of the conditions mentioned above and it is recommended that the cutter manufacturer be consulted in cases of doubt.

In certain cases, special cutters can be manufactured to give the desired gear tooth form.

Publications referred to

This standard makes reference to the following British Standard and International Standards:

BS 436, Spur and helical gears.

BS 436-2, Basic rack form, modules and accuracy (1 to 50 metric module).

BS 2519, Glossary of terms for toothed gearing (under revision).

ISO 53, Basic rack of cylindrical gears for general engineering.

ISO 54, Modules and diametral pitches of cylindrical gears for general engineering and heavy engineering.

ISO/R 701, International gear notation. Symbols for geometrical data.

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